Natural Refrigerant Training Summit

Building a Sustainable Workforce

CO2 Transcritical Booster System Operation

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NORTH AMERICAN
Sustainable Refrigeration Council

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Sustainable Refrigeration Council

CO2 Confidence

Delivering the future of CO_2 refrigeration.

CO2 Transcritical Booster System Operation with working unit

NASRC Natural Refrigerant Summit

April 4-6, 2023, Irwindale, CA

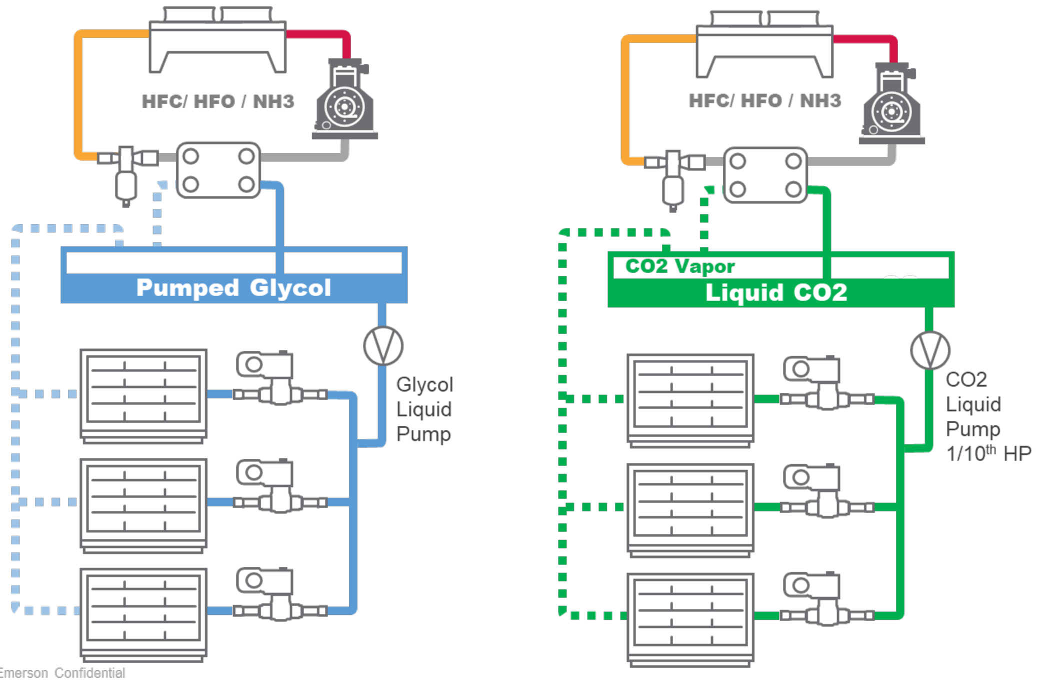
EMERSON

Agenda:

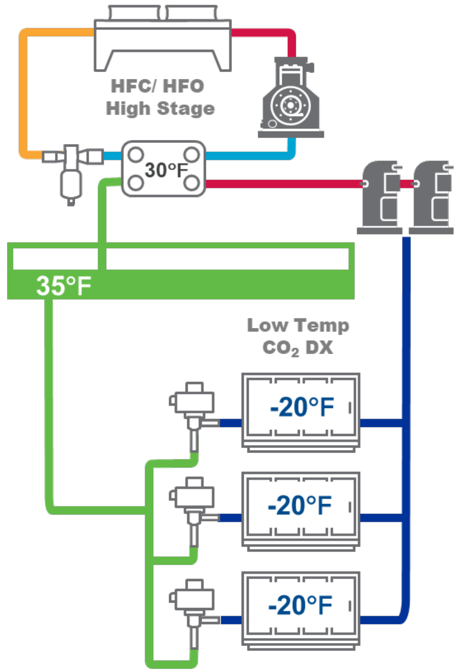
1. CO_2 as a Refrigerant
2. Safety and Handling
3. Top 10 Difference Between HCF & CO_2 Systems
4. CO_2 Unit Walk Around

Types of CO_2 Systems

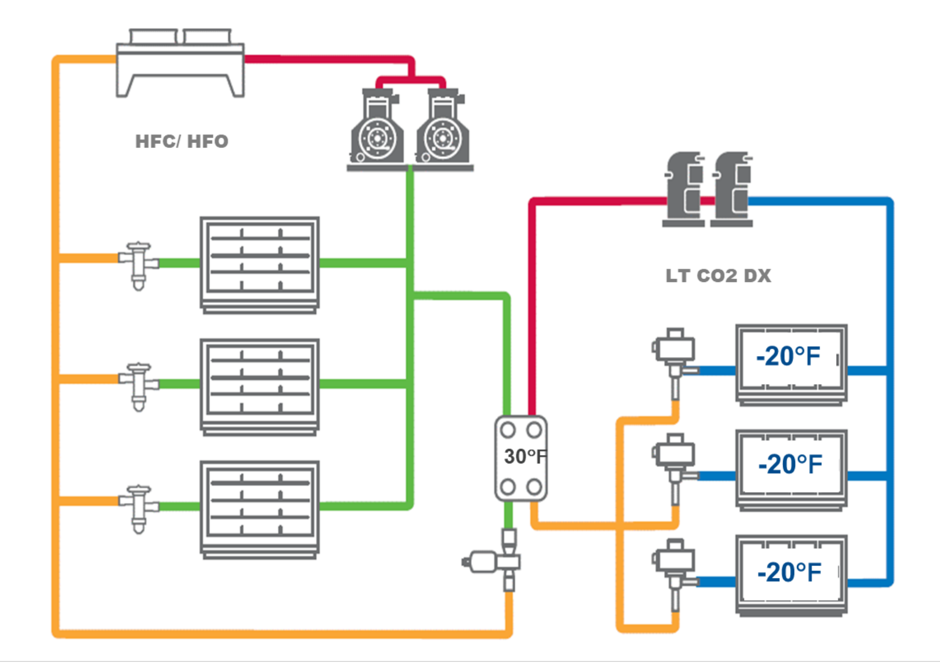
Pumped Secondary



Simple Cascade



Retail Cascade



CO_2 (R-744) as a Refrigerant

R-744 vs HCFC/HFC

	R-744	HFC / HCFC	Impact on R-744 Systems
Global Warming Potential	1	1300 to 4000	Future Proof
Ozone Depleting Potential	0	0 for HFC / High for HCFC	Future Proof
Saturation Pressures	Higher	Lower	Additional Safety Design
Operating Pressures	Higher	Lower	Specialized Components
Standstill Pressures	Higher	Lower	Relief Valves/Tanks/ etc..
(Power Outages)	Rapid Pressure Rise	Lower	Pressure Relief Venting
Inert Gas	Yes	Yes	Copper may be used
Flammability	A1	A1	Not Flammable
Toxicity	No	No	Asphyxiate in High Concentrations
Odor	None	None	Leak Detection Required
Volumetric Mass Flow	Higher	Lower	Smaller Tubes & Compressors
Heat Transfer	Higher	Lower	Better Thermal Efficiency
High Ambient Performance	Lower	Higher	System Design to Compensate
Low Ambient Performance	Good	Good	Subcritical Cascade Favorable
Cost per Pound	Low	Higher	Economical
Complexity of Systems	Higher	Lower	Higher First Cost, Training & Experience
Adoption	Low	Higher	Higher First Cost
Legislation / Regulations	Low	Higher	Long-Term Viability

R-744 Provides Many Benefits Over HFC Options

Basic Properties of R744 with R404A Refrigerants Commonly used in the Retail Sector

Refrigerant	R744 (CO_2)	R404A
Temperature at atmospheric pressure	**-109.3°F** (-78.5°C) Temp of dry ice	-50.8°F (-46°C) (Saturation temp.)
Critical temperature	**87.8°F** (31°C)	161.6°F (72°C)
Critical pressure	**1056psig**	503psig
Triple point pressure	**61 psi**	0.28.9 Hg
Pressure at a saturated temperature of 20°C	**815psig**	144psig
Global warming potential	1	3922

CO_2 (R-744) - Refrigerant Safety & Handling

PPE – Personal Protective Equipment

Hazards

Odourless, Tasteless

Toxicity levels (greater than most HFCs)

Heavier than air (asphyxiation concerns)

Higher pressures (compared to HFCs)

Low temperature exposures (frost-bite)

High temperature exposures (burns)

Toxicity Levels of R-744 (CO_2)

ppm of CO_2	Effects
370	- atmosphere concentration
5,000	- max. 8-hour limit exposure
15,000	- max. 10-minute limit exposure
30,000	- discomfort, breathing concerns, headache, dizziness, may have a taste
100,000	- loss of consciousness, death
300,000	- ***quick death!***

5000 ppm = 0.5% concentration

TLV for **CO_2** = 5000 PPM
TLV for **R-404A** = 1000 ppm

Temperature Exposure

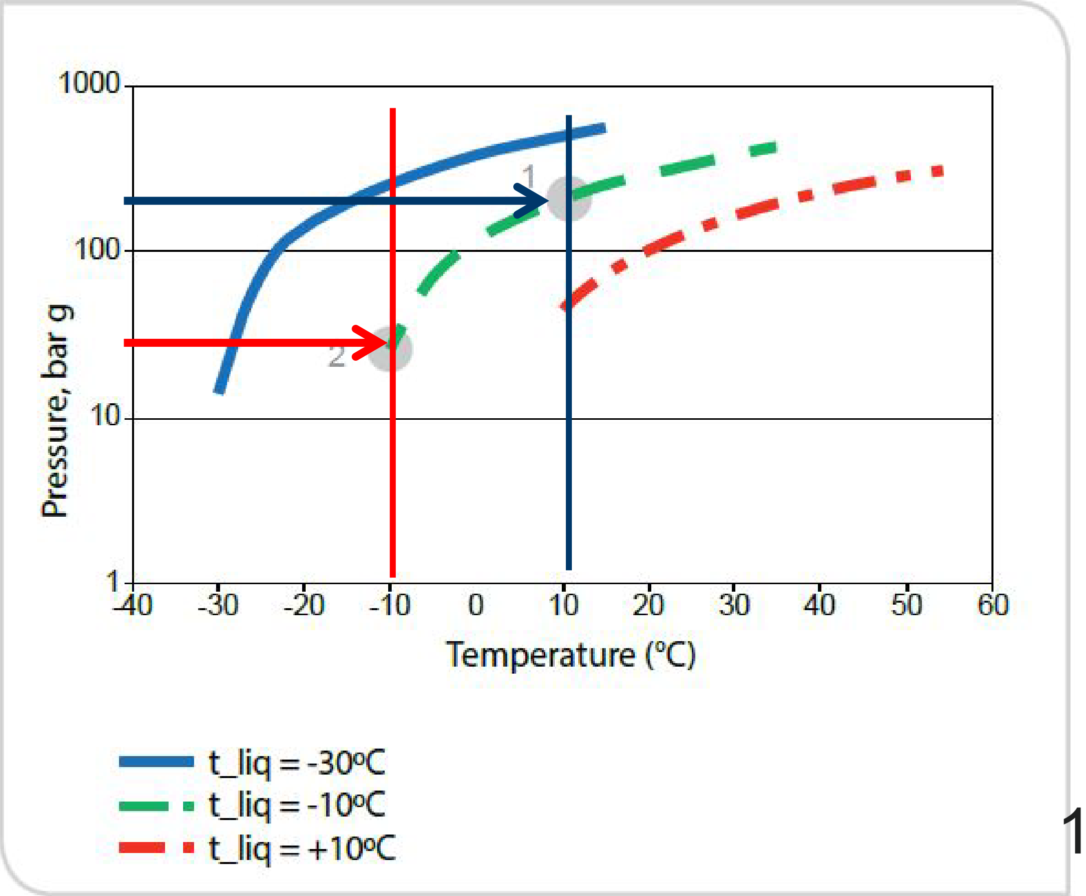
Boiling Temperature of CO_2

-78.5 °C (-109.3 °F)

As with all refrigerants frostbite and burns are a concern

Due to the extreme cold temperatures of CO_2, extra precautions are required

Liquid Expansion



Trapped Liquid Starting at

-10 C (14°F) 368psig (25 bar g)

Undergoing a Temperature Rise of

20K (or 36F)

Pressure Increases from

368 psig to 3268 psig =

2900 psig Increase (200 bar g)

Rule of Thumb

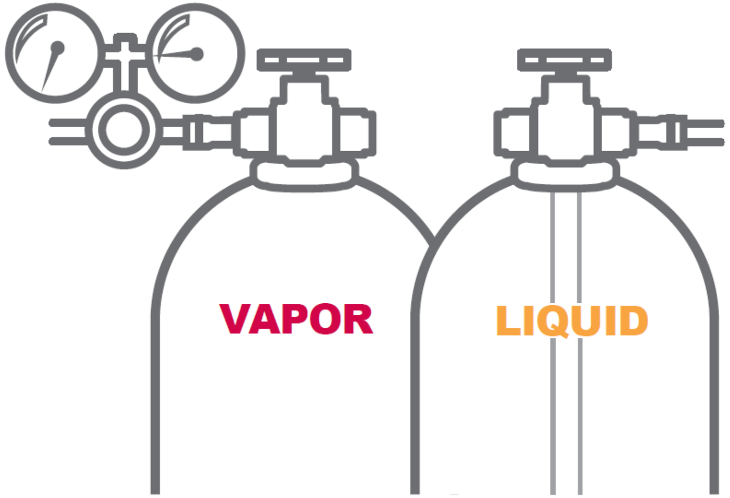
1°K (1.8°F) increase

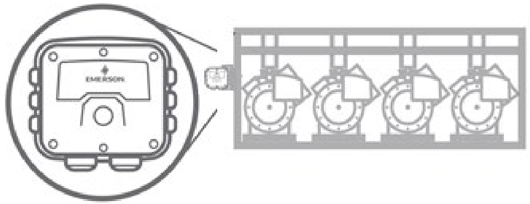
10 bar increase or 145 psig

R744 (CO_2) Cylinder

Dip Tube Line Designating Liquid Tank

Grade 4 = 99.99% Pure
Refrigeration Grade





Relative vapour density Vs Air is 1.52
It collects low areas

Cylinder standstill pressures are rated to 1800 psi at 68°F (20°C)

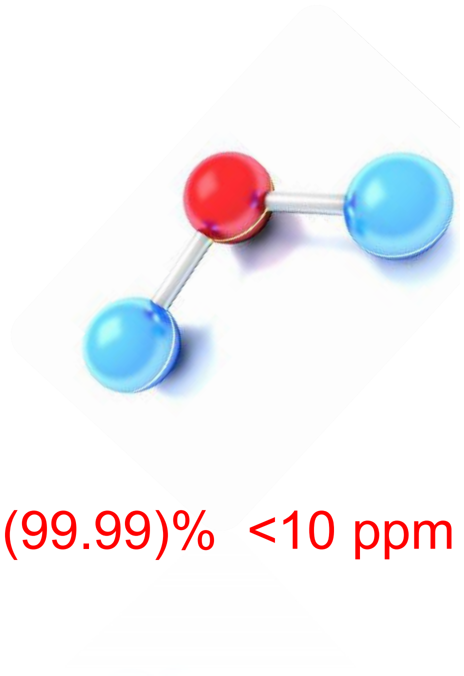
Cylinder Adapters

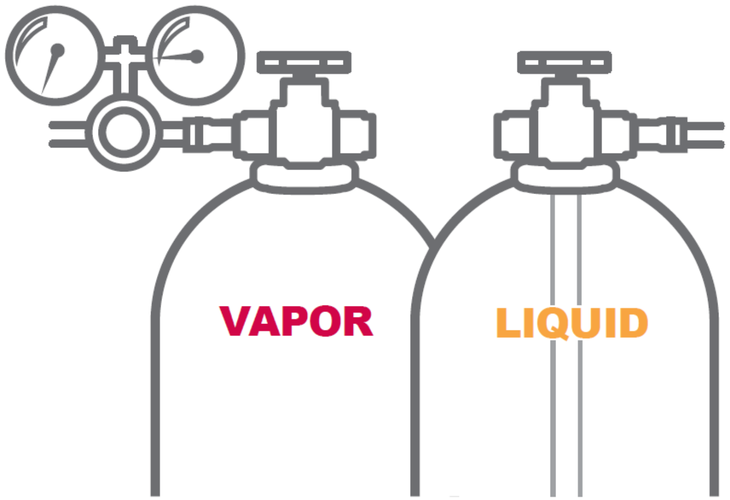
Service connector/adaptor is a CGA 320

Refrigerant Grade (Quality)

The difference is moisture content

- SFC (with helium) (99.999%) <1ppm
- SFC (without helium) (99.999%) <1ppm
- Research (99.999%) <0.5 ppm
- Refrigeration Grade, Coleman, Anaerobic (99.99)% <10 ppm
- Bone dry, Commercial (99.8 %) (liquid)





TOP 10 DIFFERENCES BETWEEN CO_2 AND HFC SYSTEM DESIGNS

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KEY DESIGN CONSIDERATIONS OF CO_2 TRANSCRITICAL BOOSTER SYSTEMS COMPARED TO TRADITIONAL HFC SYSTEMS

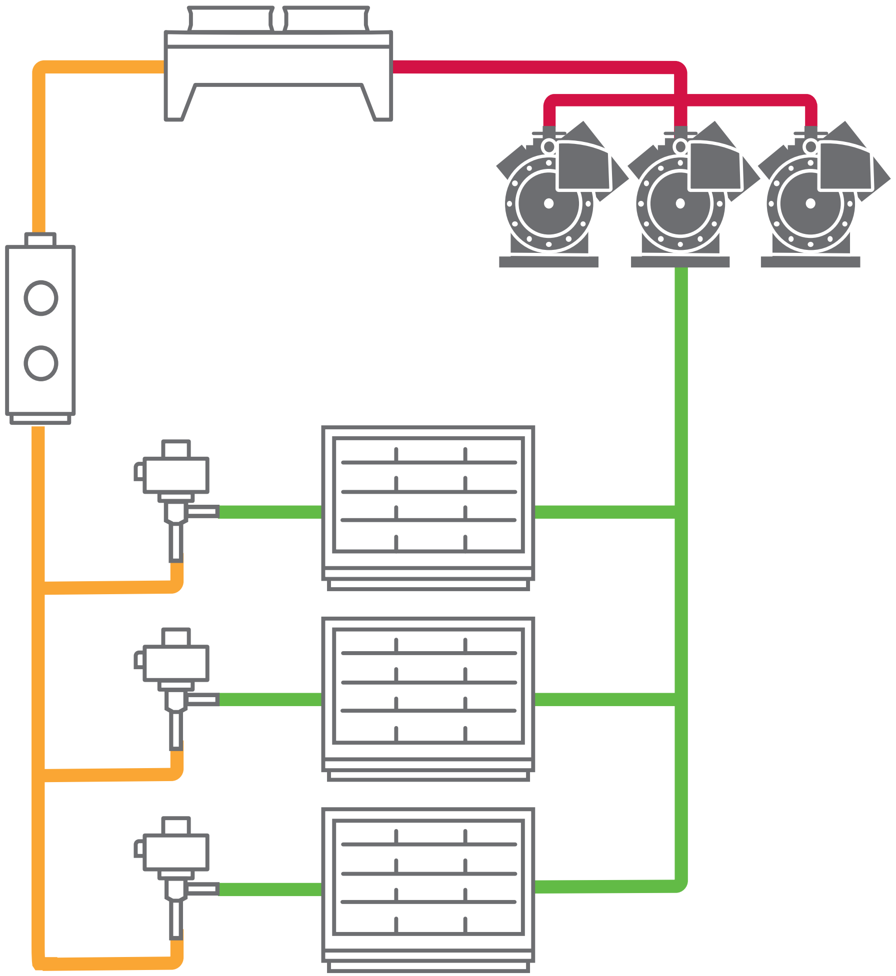
CO_2 transcritical booster (TCB) refrigeration systems have proven to be safe, reliable and efficient in tens of thousands of installations around the globe.

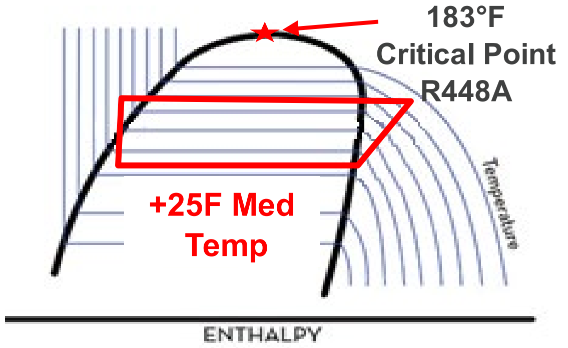
To meet sustainability targets, installations of CO_2 TCB systems are expected to rise significantly over the coming years in U.S. food retail markets.

Compared to systems based on legacy hydrofluorocarbon (HFC) refrigerants, CO_2 (refrigerant name R-744) TCB systems have unique characteristics that require specific mitigation strategies and design considerations.

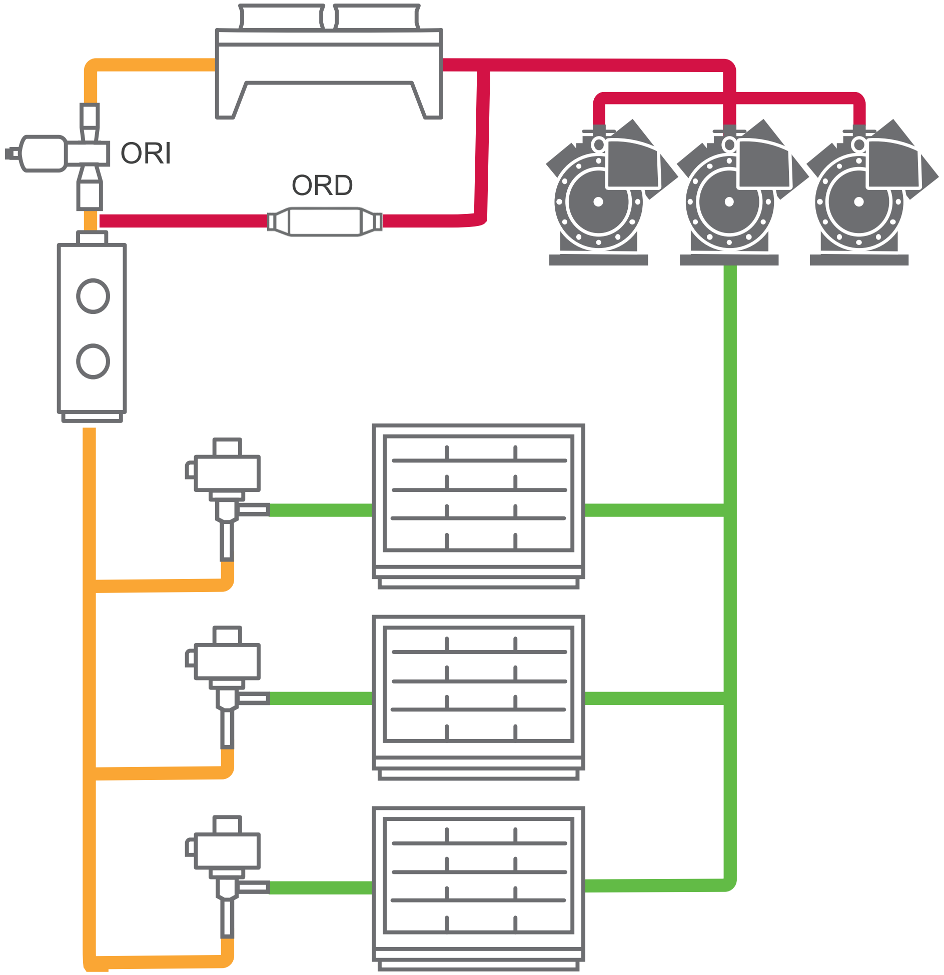
Original equipment manufacturers (OEMs), contractors and end users should begin familiarizing themselves with the key design differences and operating principles between CO_2 TCB and traditional HFC systems.

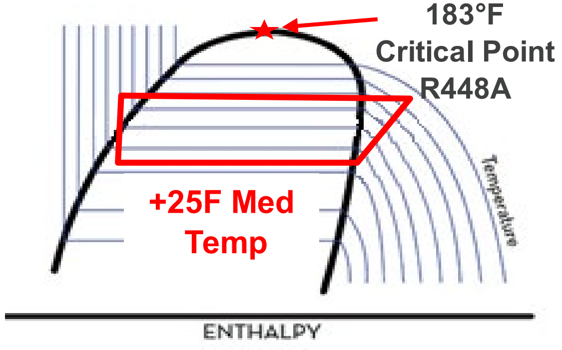
R448A basic MT System



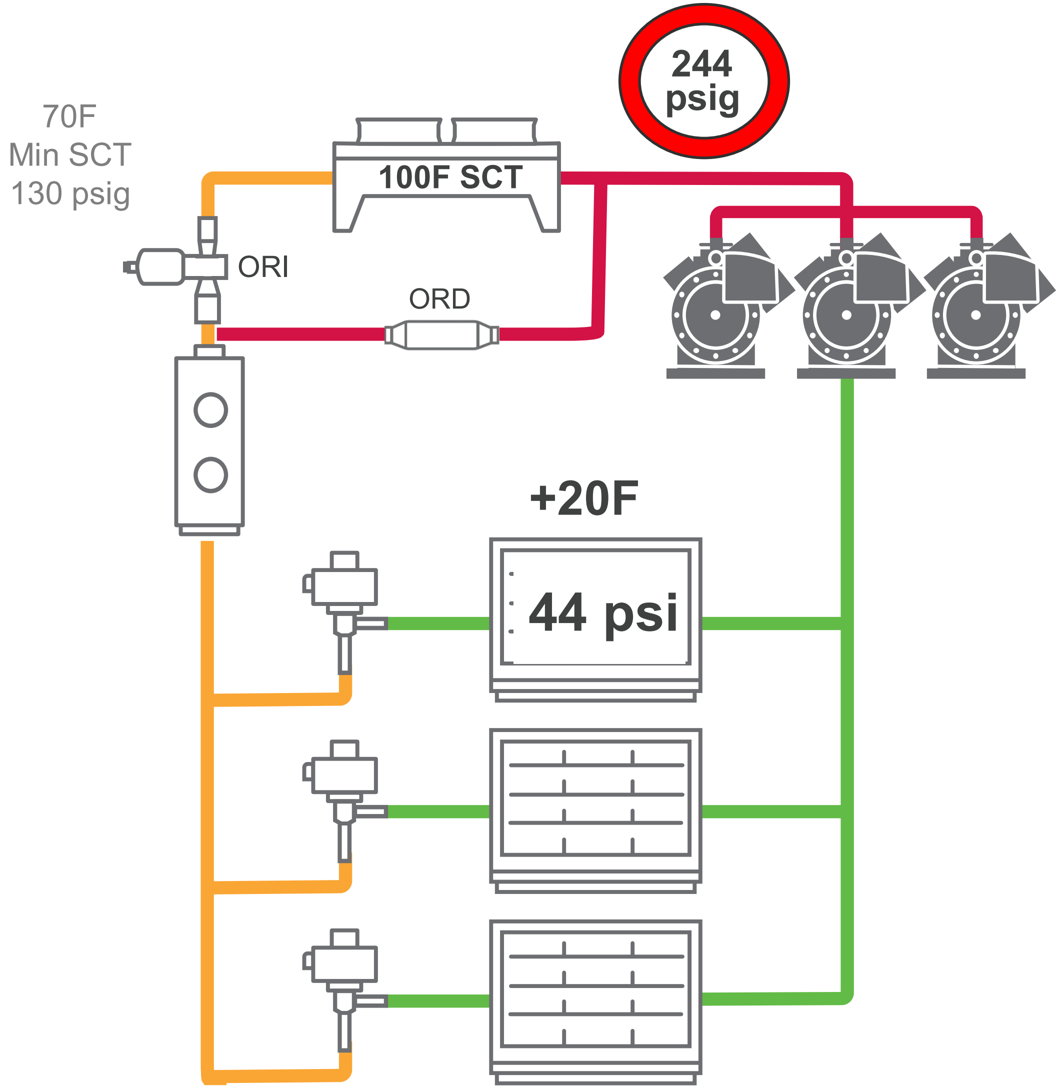


R448A With Head Pressure Controls System

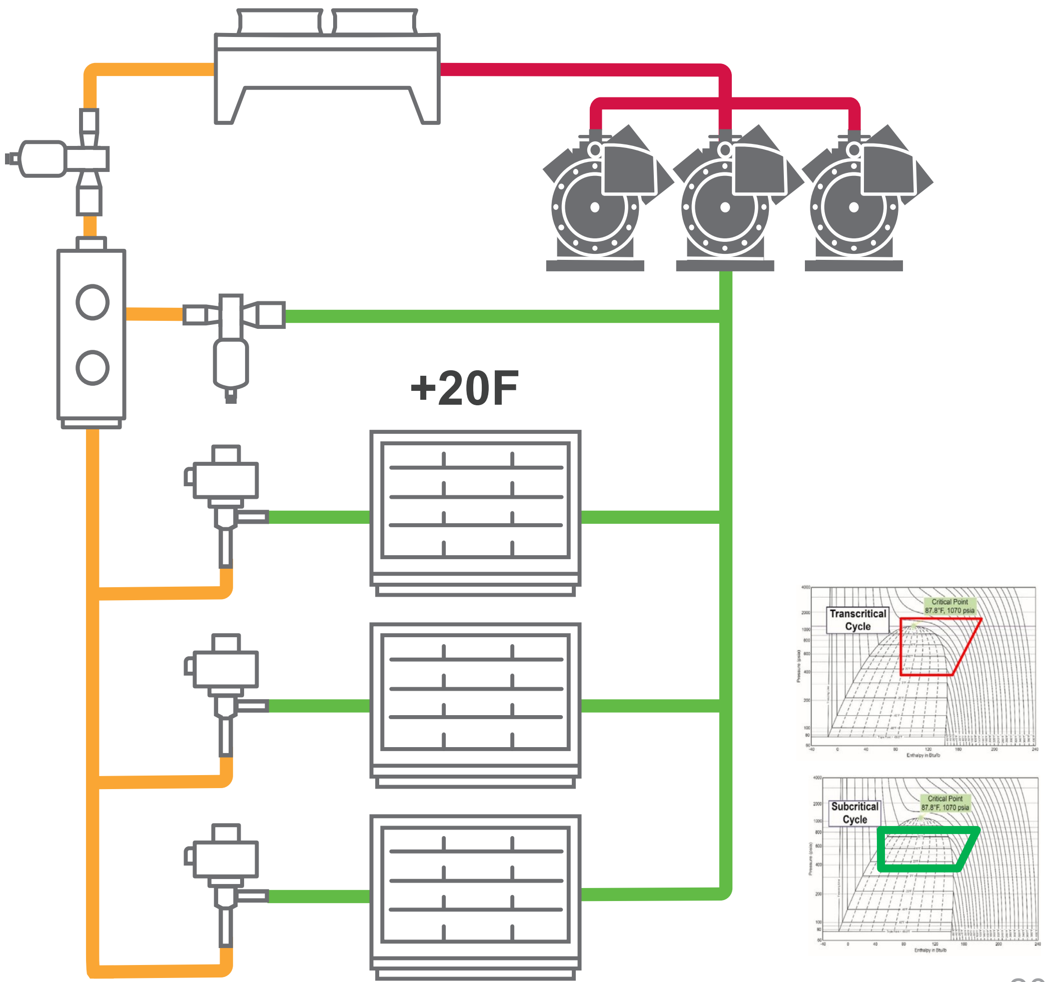




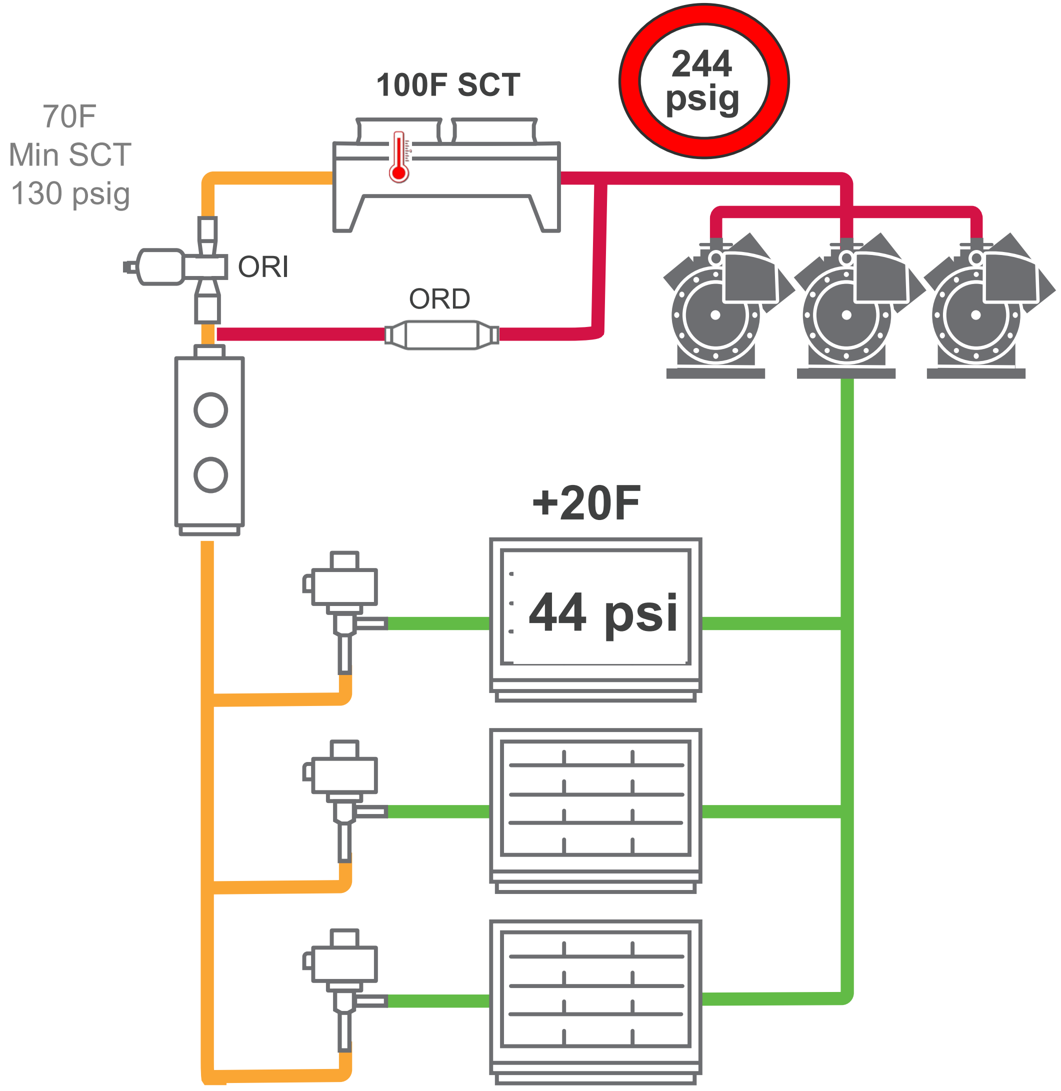
R448A With Head Pressure Controls



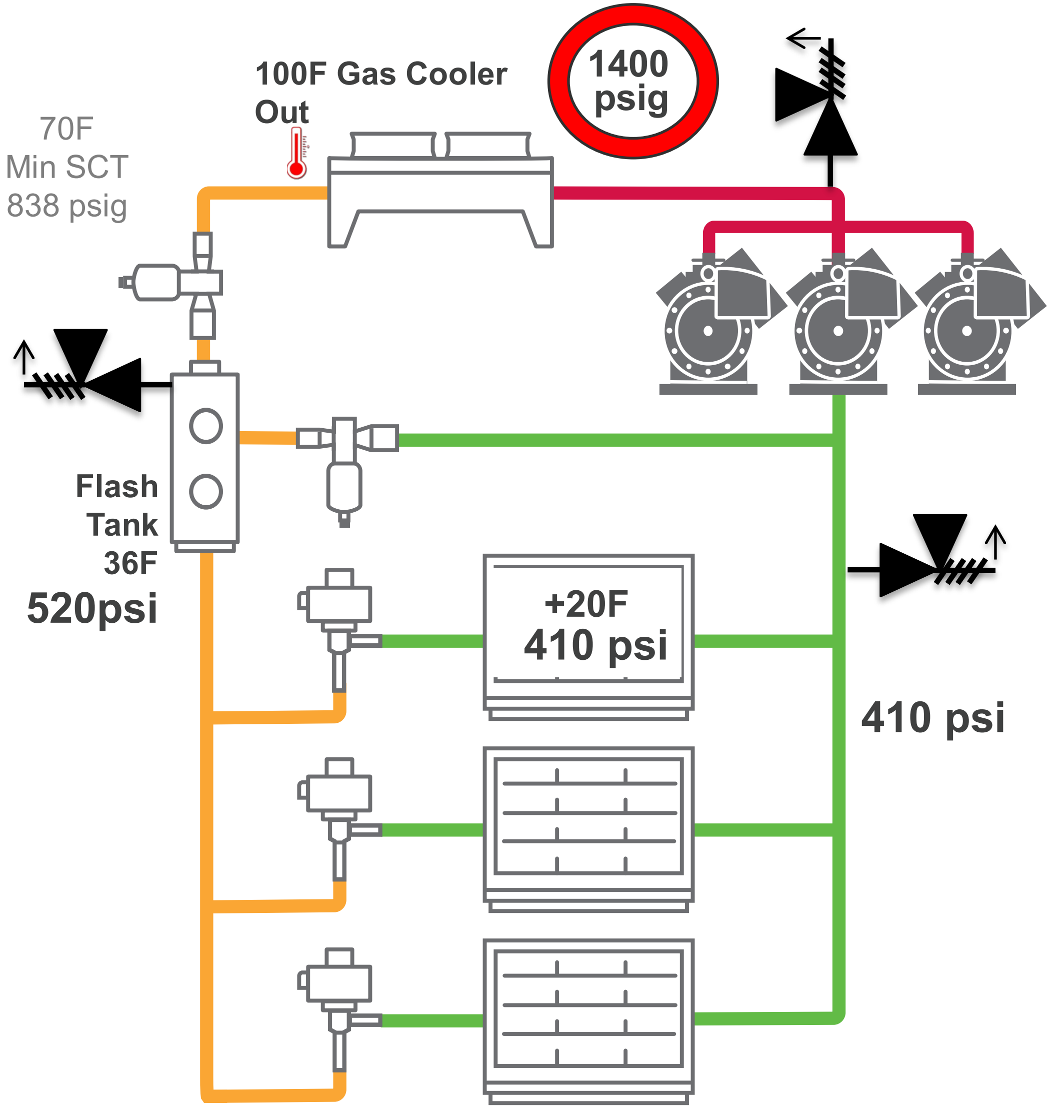
Medium Temp CO_2 Transcritical Systems



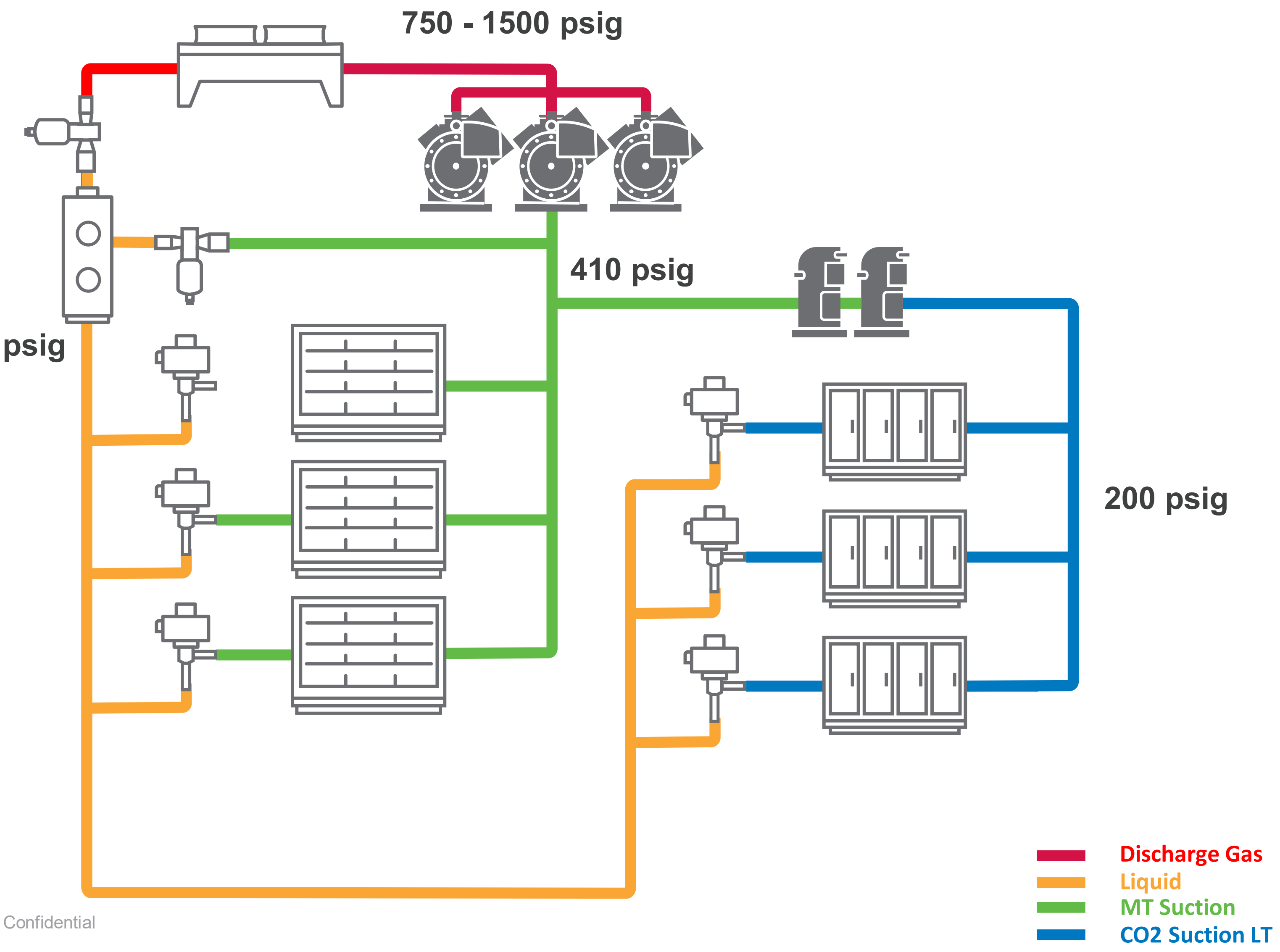
R448A With Head Pressure Controls



Medium Temp CO_2 Transcritical Systems



Types of CO_2 Systems



CO_2 Transcritical Booster (TCB)

10 Major Difference VS HFC

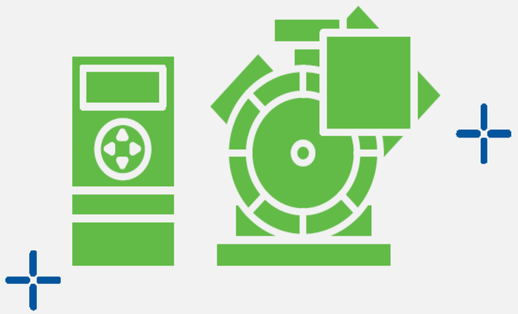
1. Booster Design
2. Gas Cooler (TC) / Condenser (SC)
3. Higher Discharge Pressure vs HFC
4. Low Critical Point CO2
5. High Pressure Valve Bypass Valve
6. Flash Tank / Receiver
7. Reliance on Controls
8. High Triple Point
9. Standstill Pressure
10. Safeties and Pressure Relief Valves

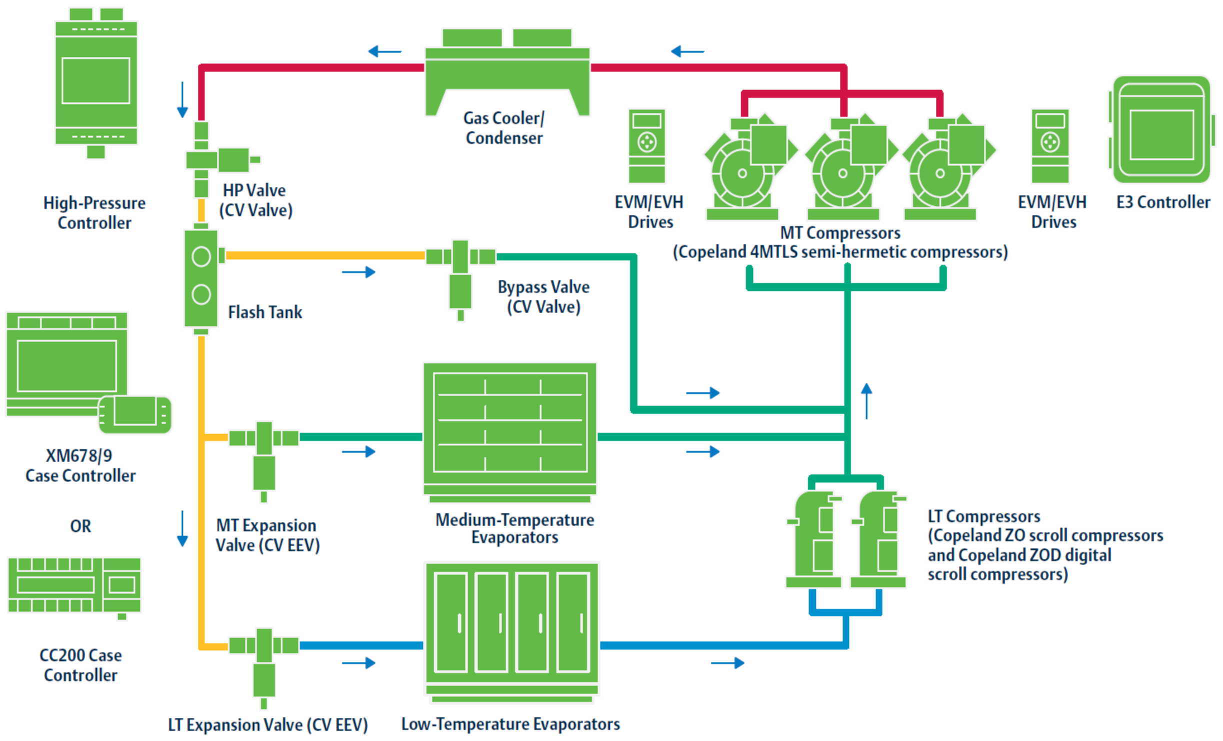
1

BOOSTER DESIGN

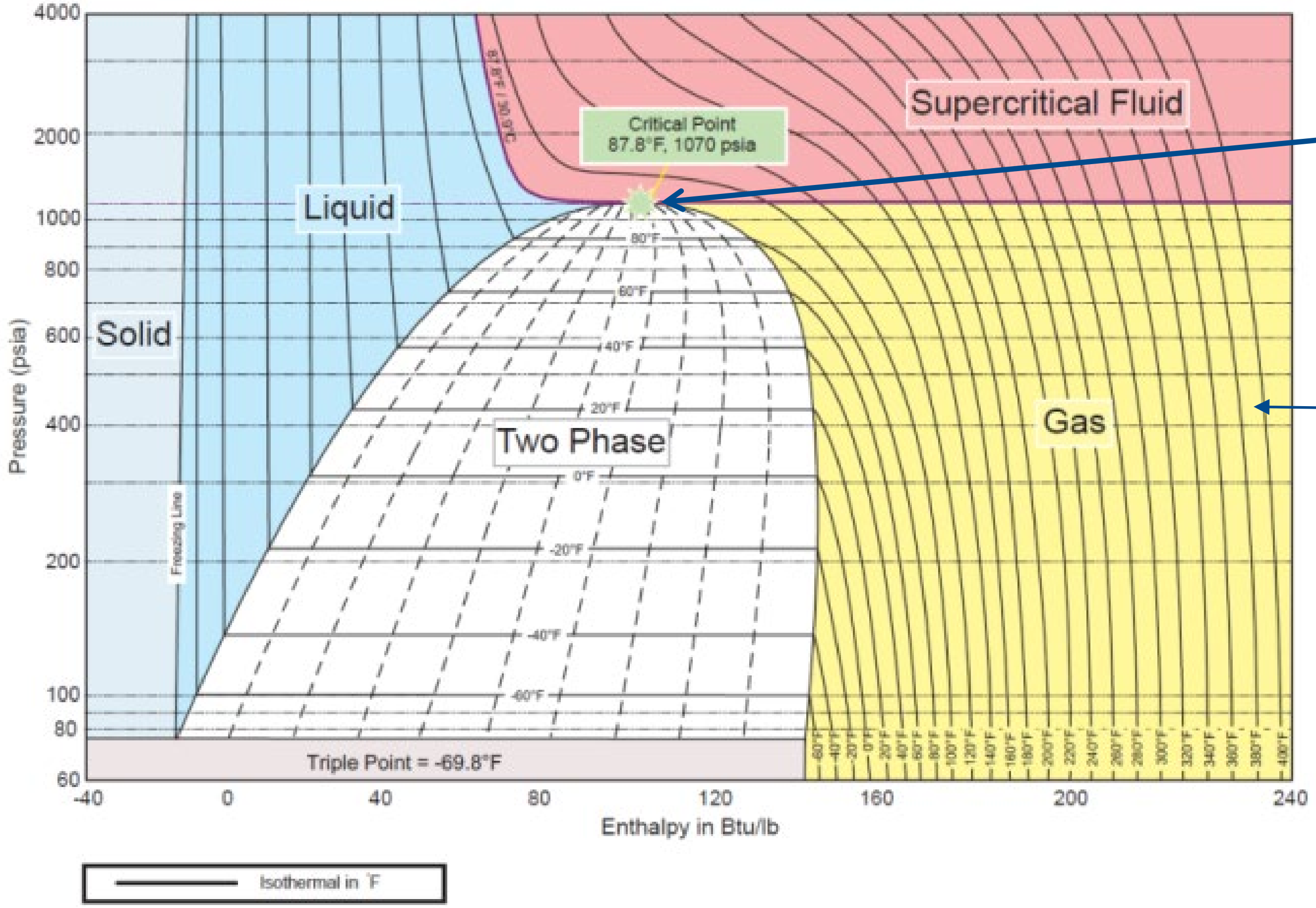
CO_2 TCB systems run solely on R-744 for both medium- (MT) and low-temperature (LT) refrigeration loads.

- They are called *booster* systems because the LT compressors do not discharge directly to the condenser, as they would in HFC systems.
- Instead, the LT compressors discharge R-744 into the MT compressors, which then boost the refrigerant to a gas cooler.



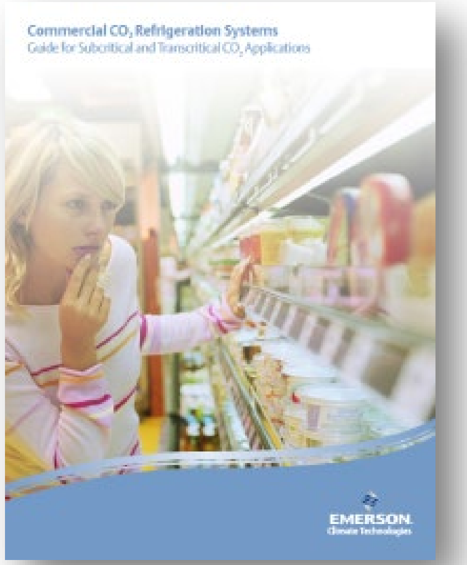


Pressure-Enthalpy Diagram, CO_2



Liquid and Gas Density Are the same ONLY at Critical Point

P. 7



https://www.youtube.com/watch?v=-gCTKteN5Y4

Transcritical Systems Can "Transition" from Subcritical to Supercritical

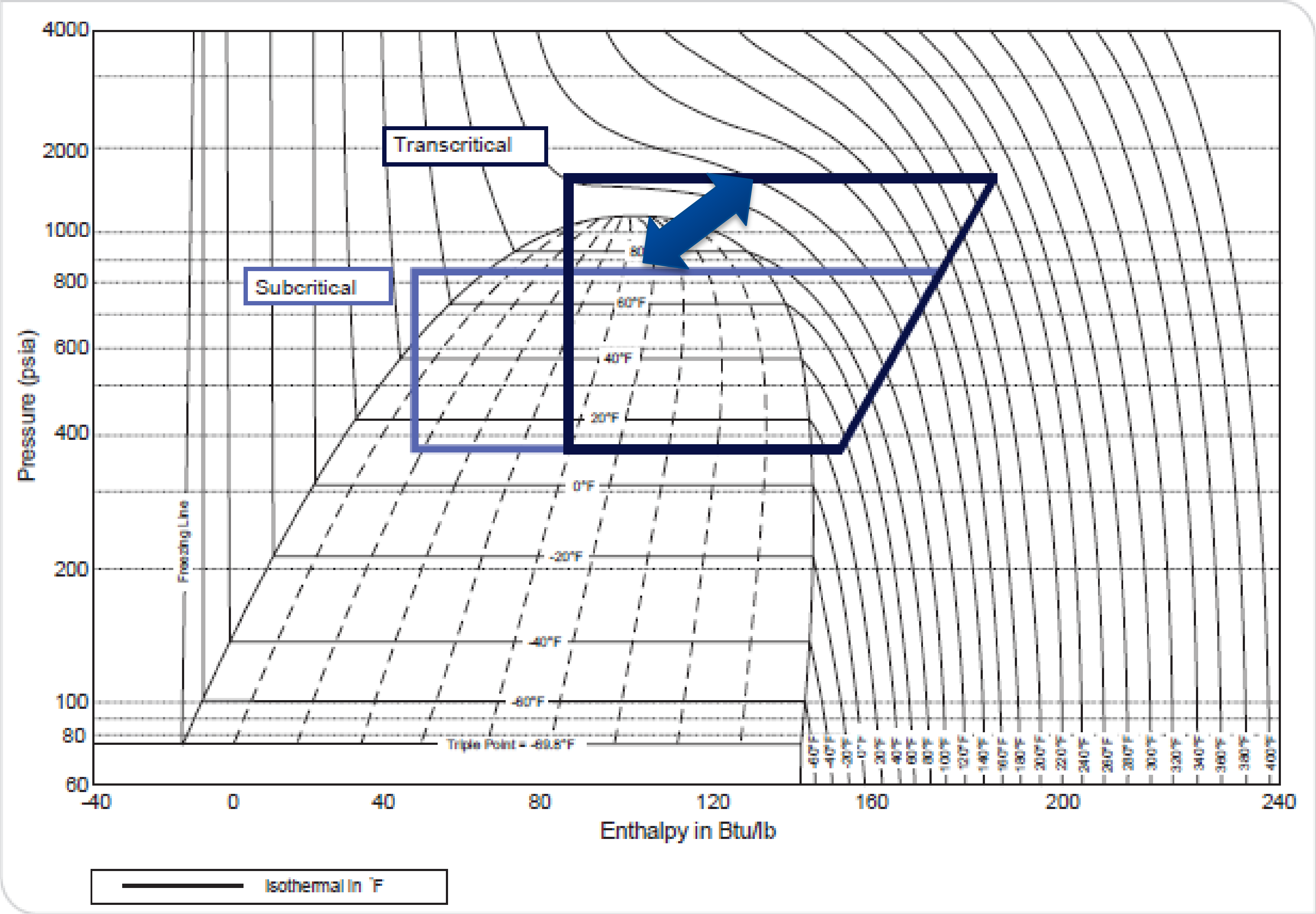
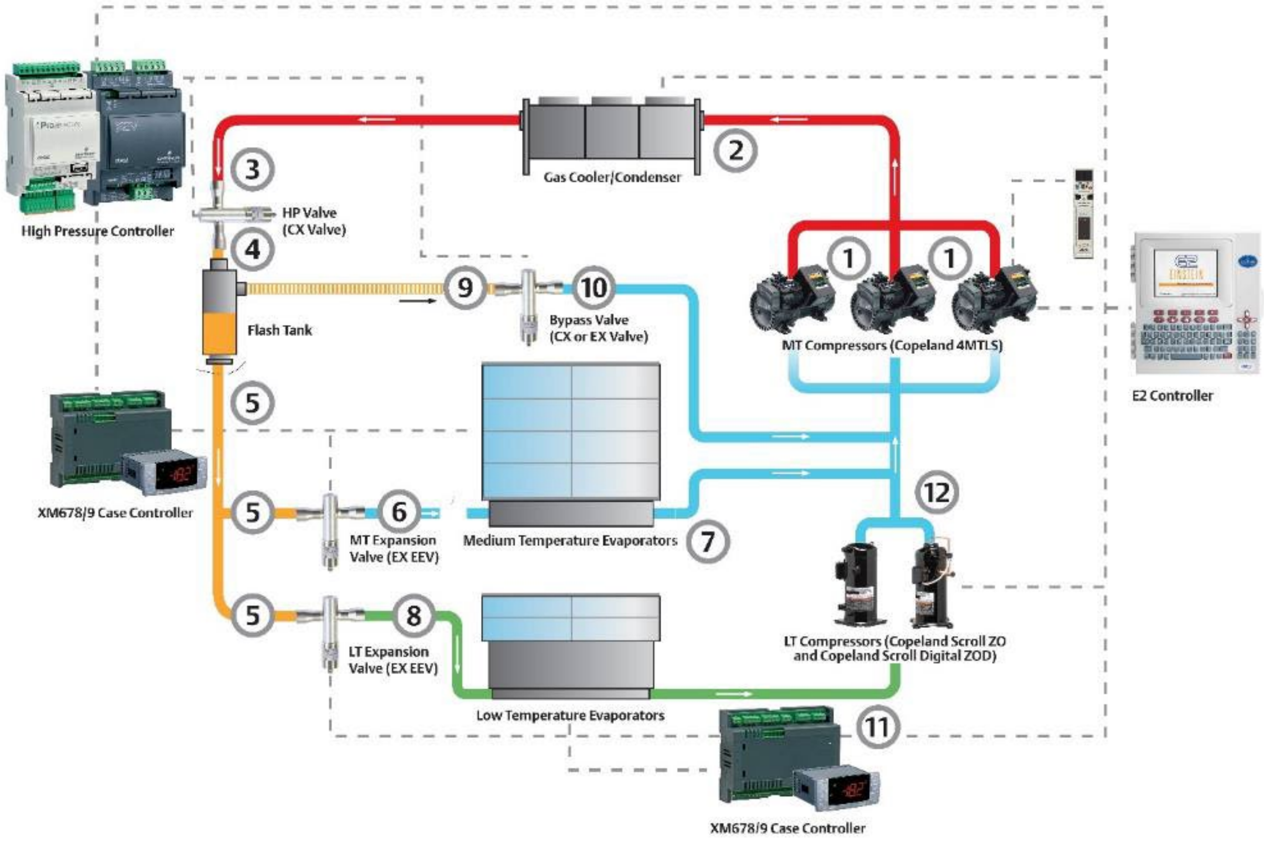
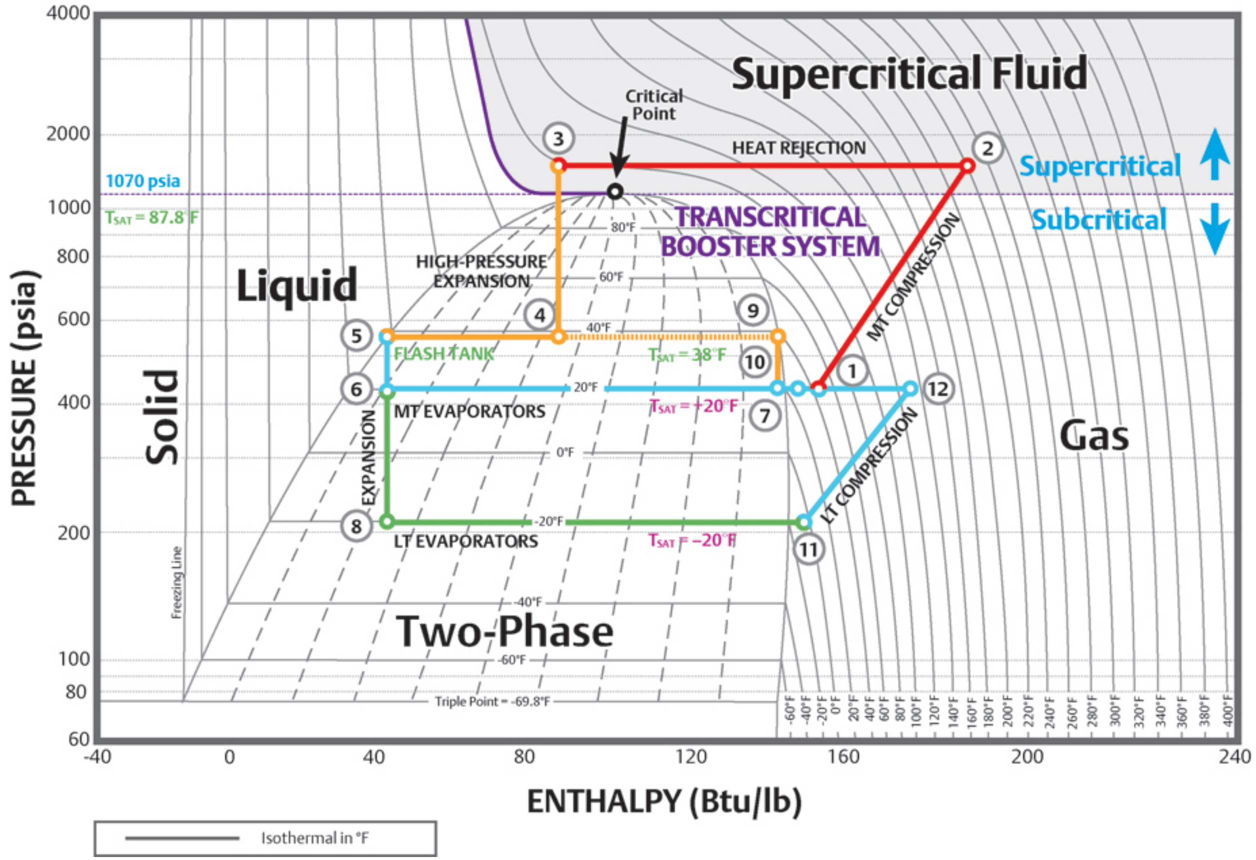


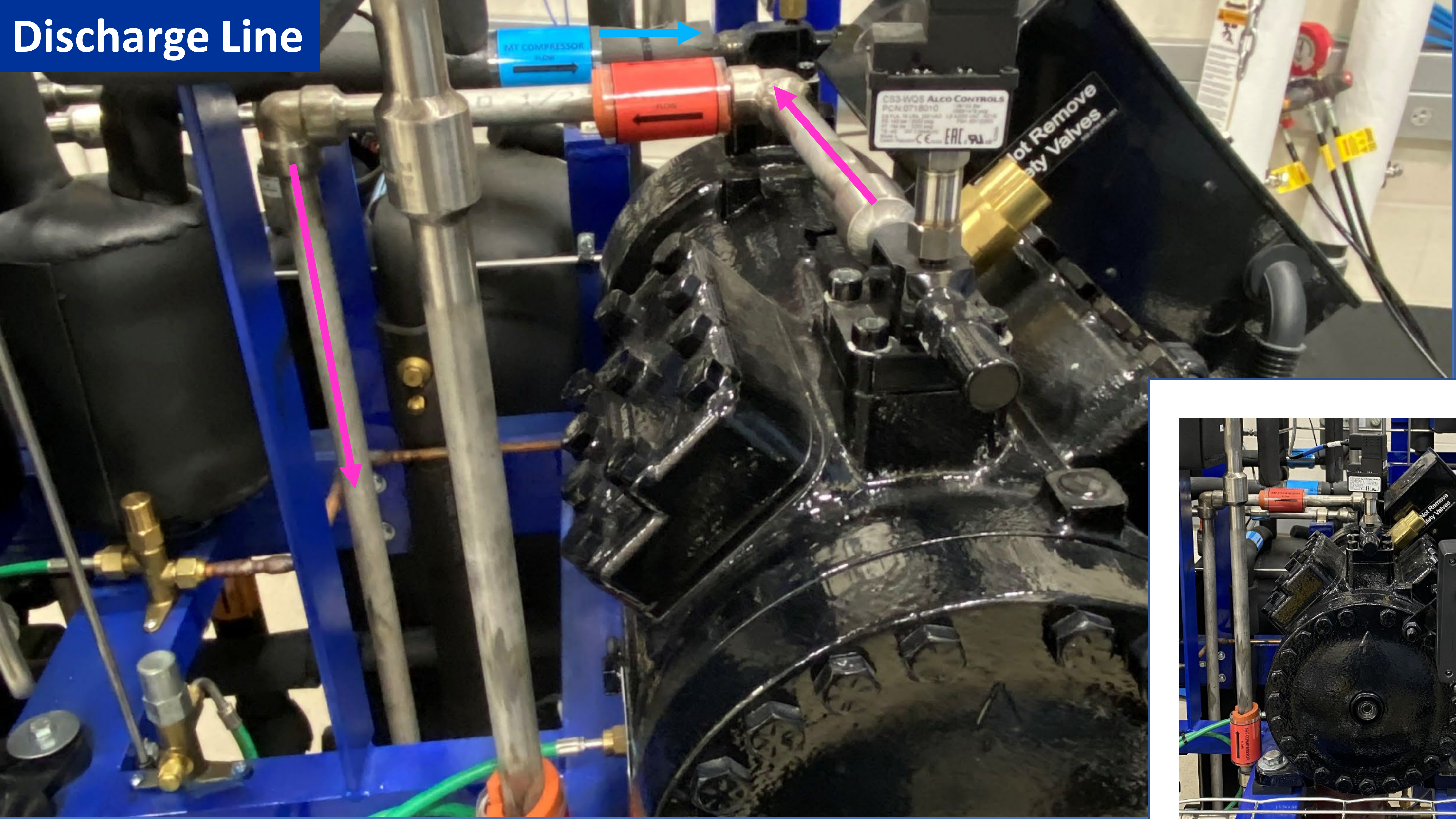
Figure 6. R744 pressure enthalpy chart showing subcritical and transcritical systems

Transcritical CO_2 Booster System - Animation



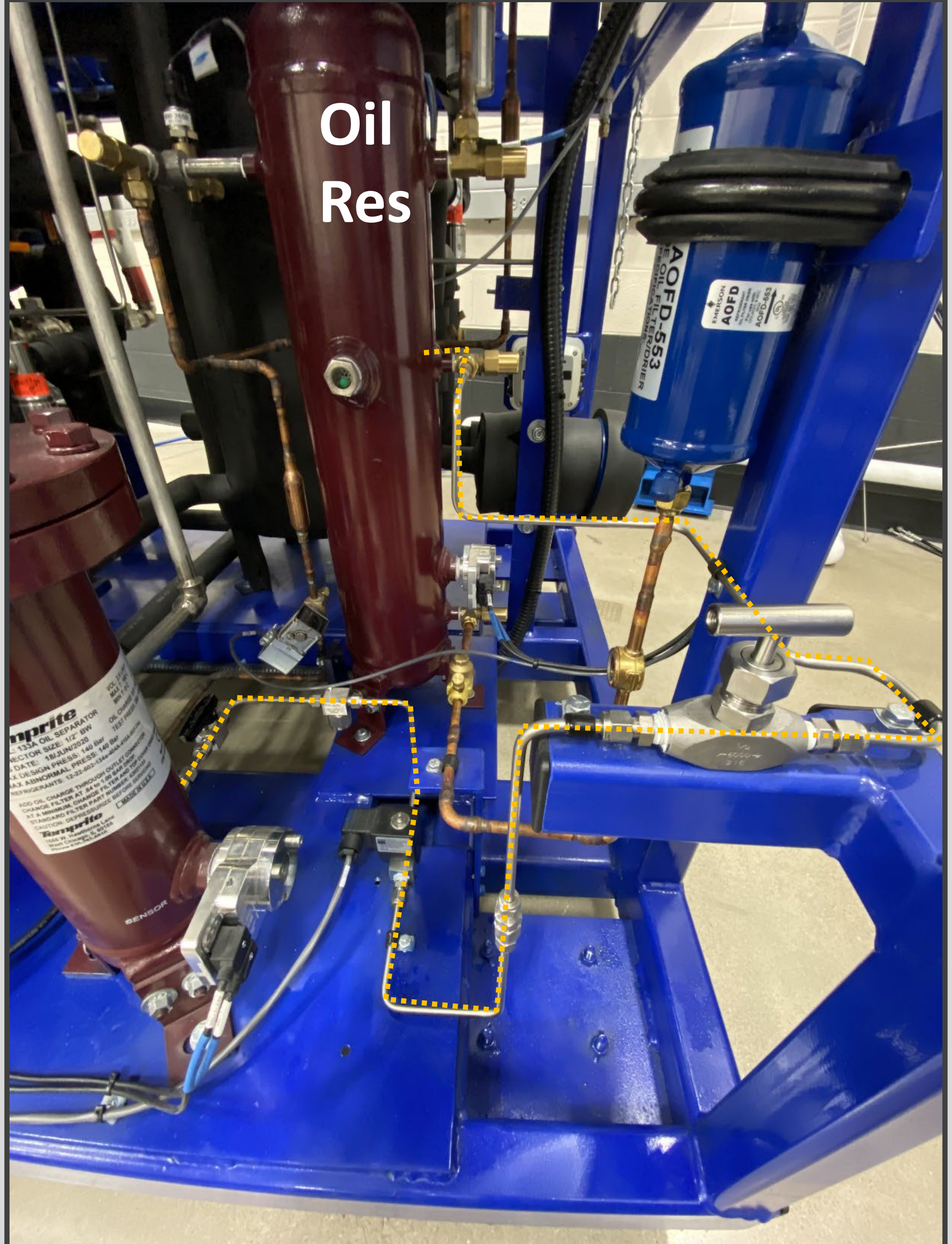


Discharge Line

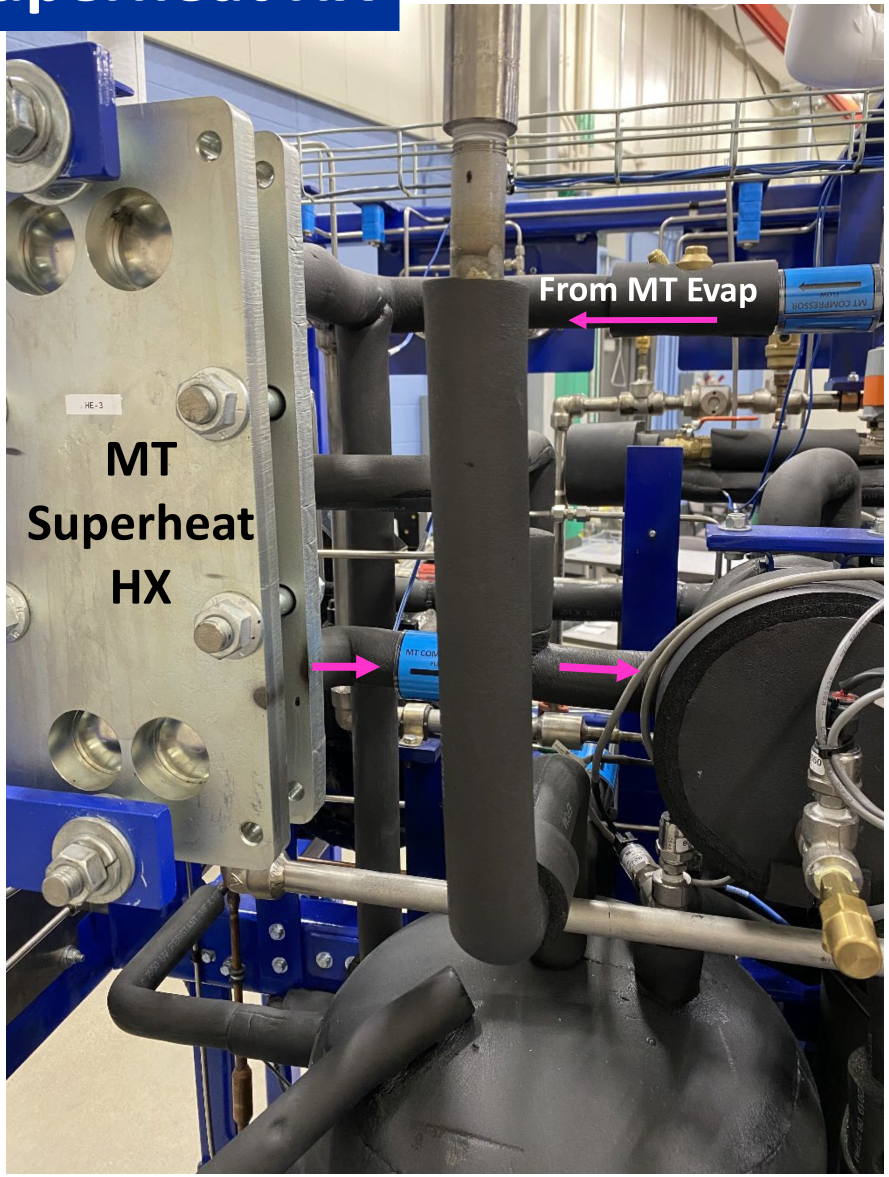


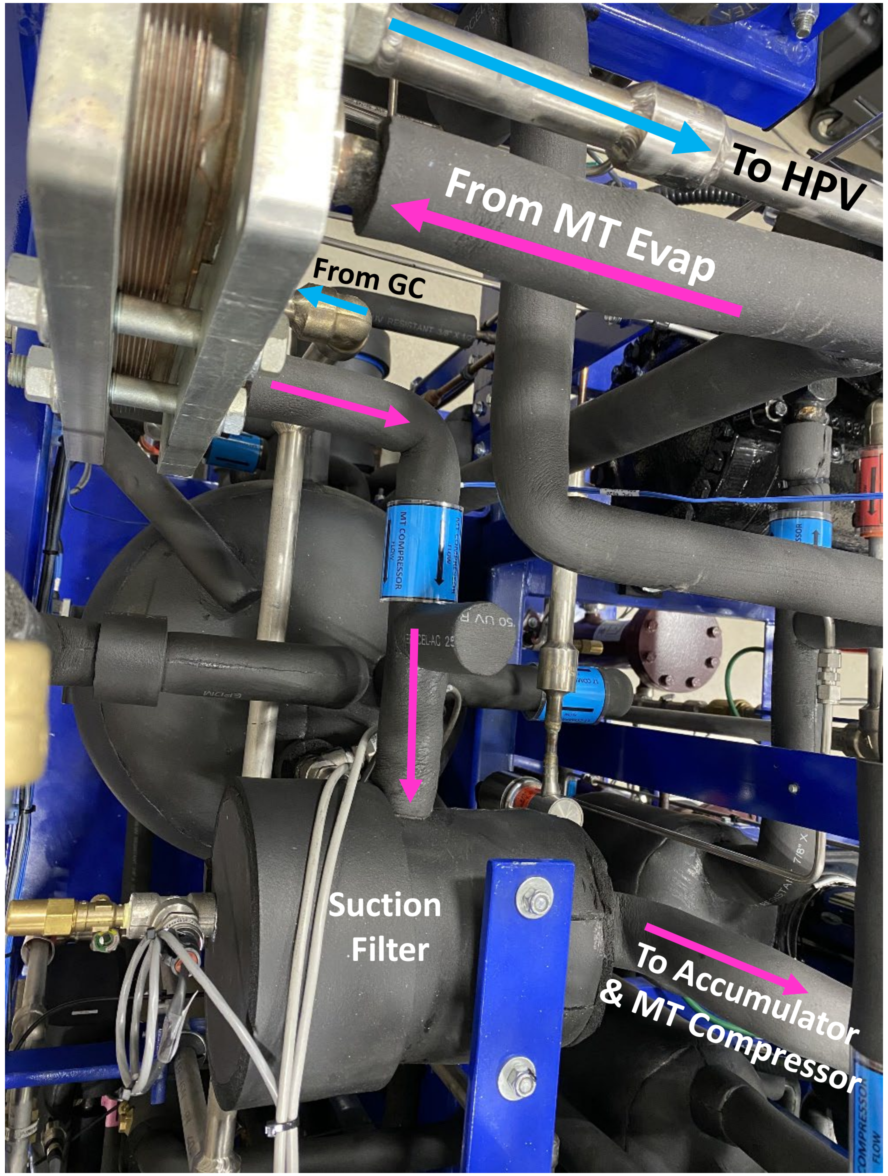
Copeland 4MTLS40KE-FSC-C00

Oil Separator



MT Superheat HX





Design Differences Between HFC and CO_2 Transcritical Compressor

Compressors of same Capacity

Discus 4DH

Copeland™ brand products

CO2 4MTLS15

Copeland™ brand products

Stator Cover

Discus 4DH

005-1832-00

12 bolts

CO2 4MTLS15

505-1218-00

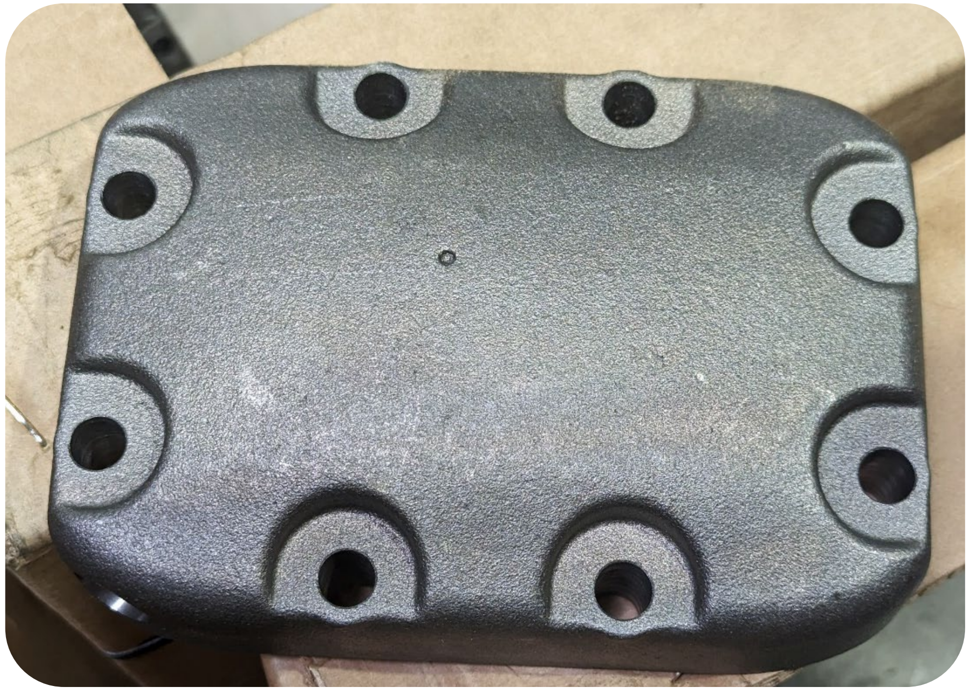
27 bolts

Copeland™
brand products

Head Covers

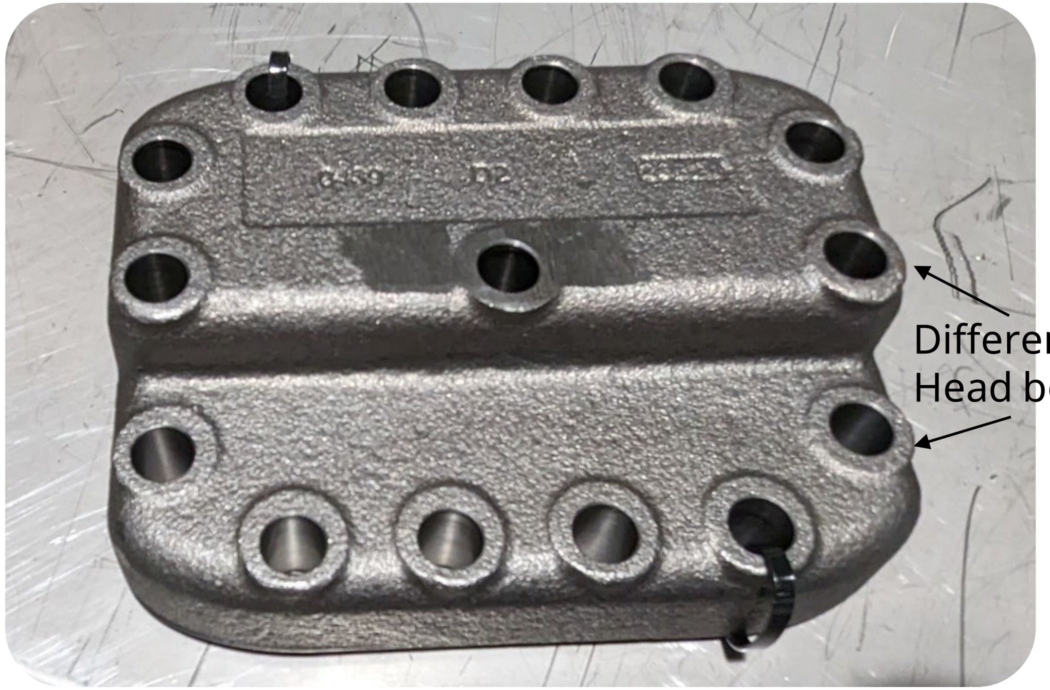
Discus 4DH

002-0433-00



CO2 4MTLS15

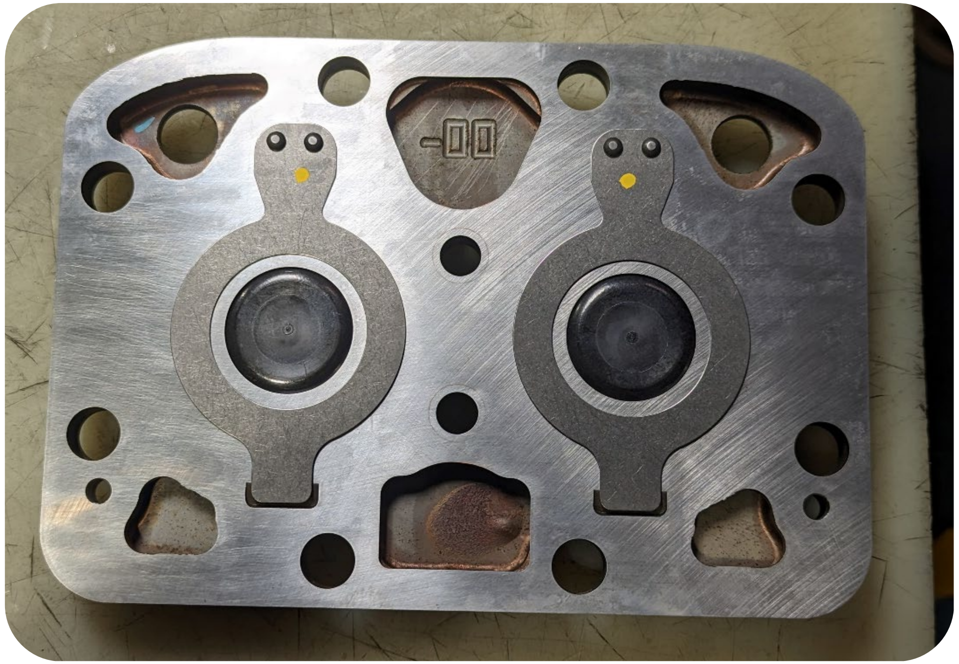
002-0439-00



Different Length
Head bolts

Valve Plates

Discus - 4DH



503-2032-00

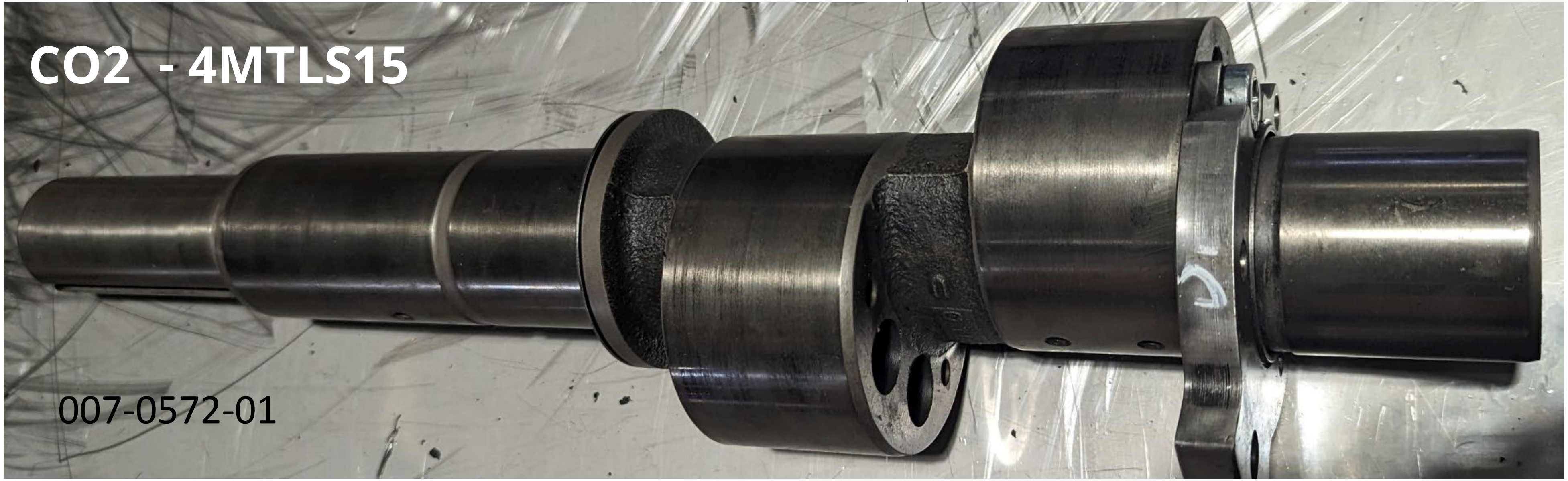
CO2 - 4MTLS15

503-1075-00

Note: Discus "puck" design vs. traditional Reed

Copeland™ brand products

Crankshaft



Copeland™
brand products

Connecting Rods

Discus - 4DH

Discus
Large Diameter 47mm
Small Diameter 19mm

CO2 - 4MTLS15

CO2
Large Diameter 82mm
Small Diameter 25mm

DU Bearings pressed in each hole

Copeland™ brand products

Wrist Pins

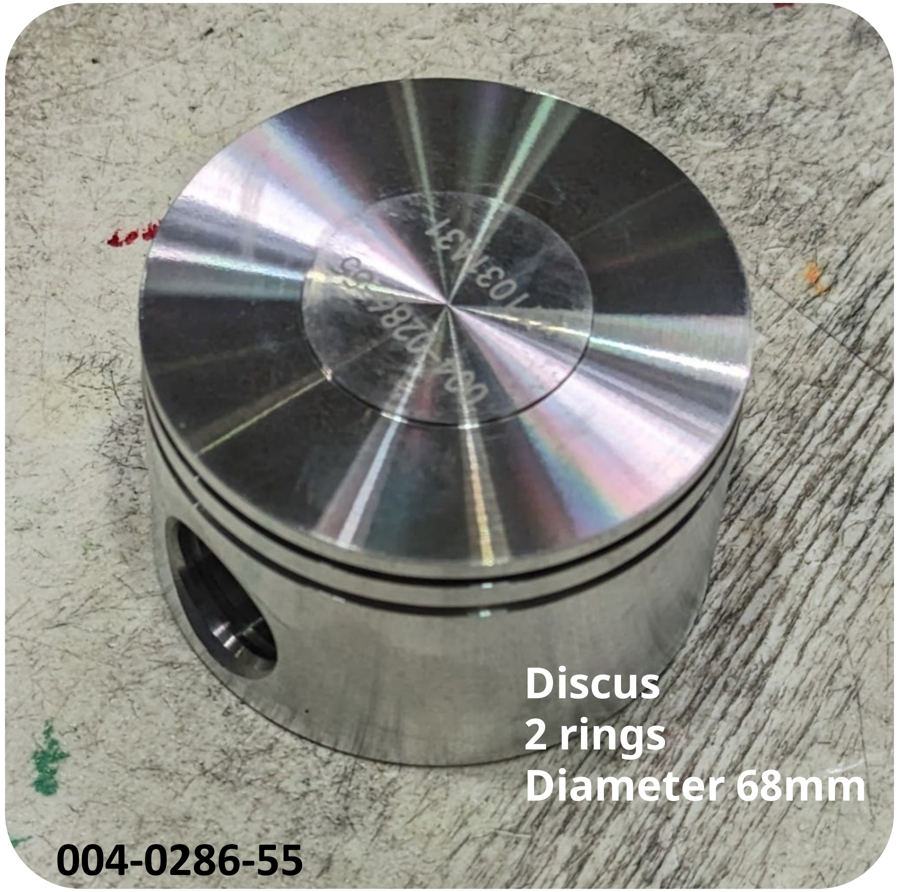
Discus - 4DH

CO2 - 4MTLS15

Copeland™
brand products

Pistons

Discus - 4DH



CO2 - 4MTLS15

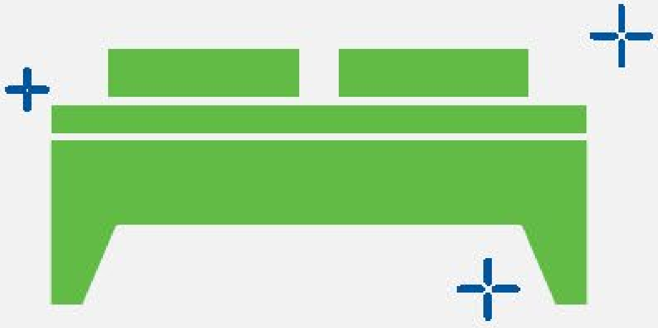
Copeland™
brand products

2

GAS COOLER

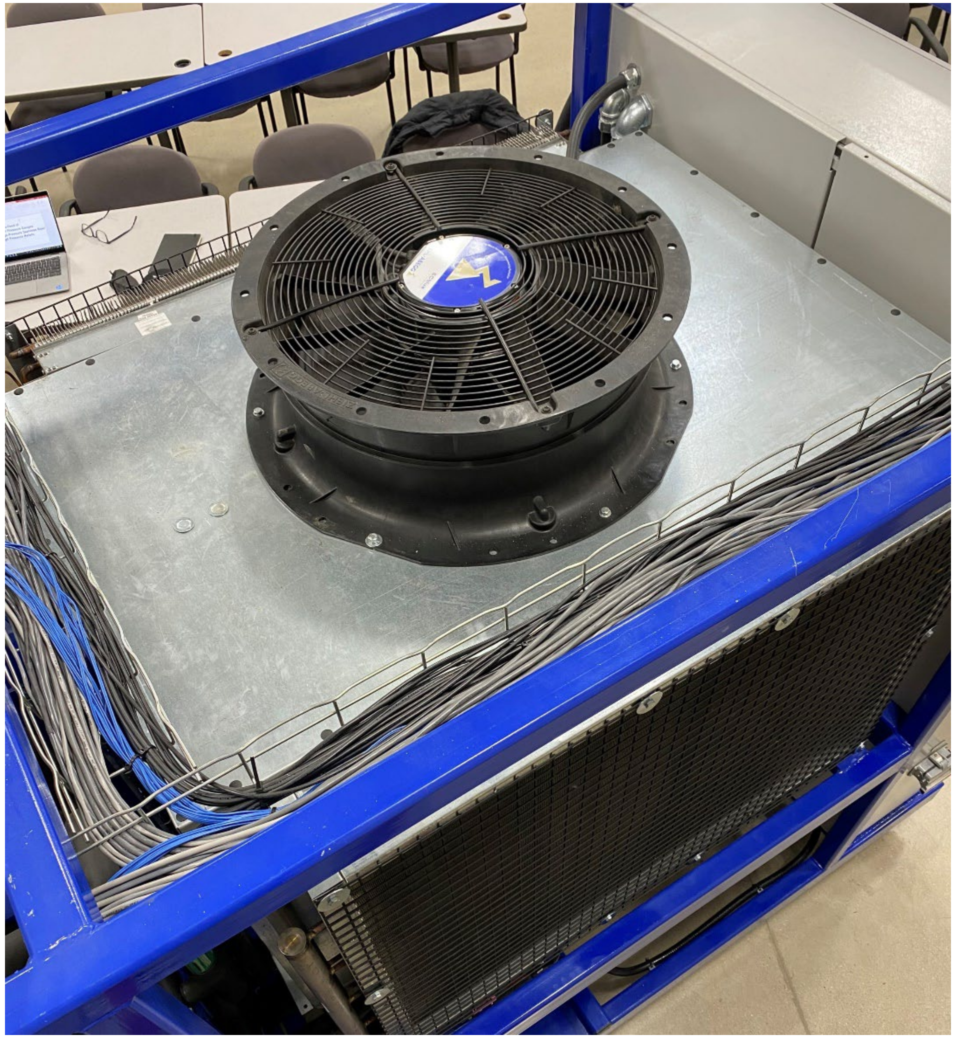
The gas cooler (aka condenser), typically located on the roof, is integral to a CO_2 TCB system's design.

- Must be sized to handle the system's total heat of rejection from MT compressors at an installation location's design conditions
- Typically designed with variable speed fan motor control
- Can include adiabatic cooling pads to improve system efficiencies in warm ambient climates



GAS COOLER/CONDENSER

Gas Cooler (Training Unit)



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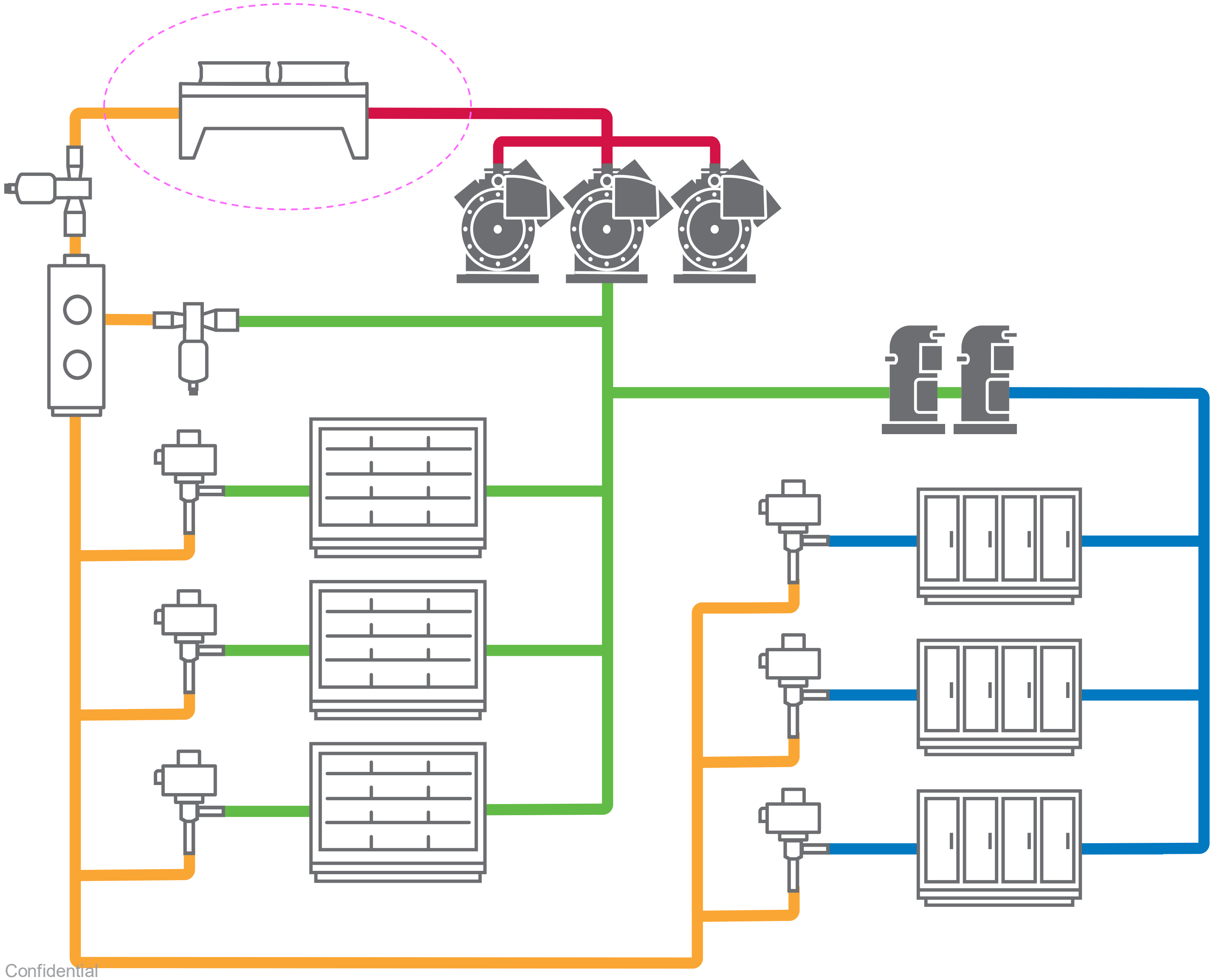
CO_2 Transcritical Booster Systems Gas Cooler



130 bar (1885psi) Design
Variable Speed Fans (ECM)

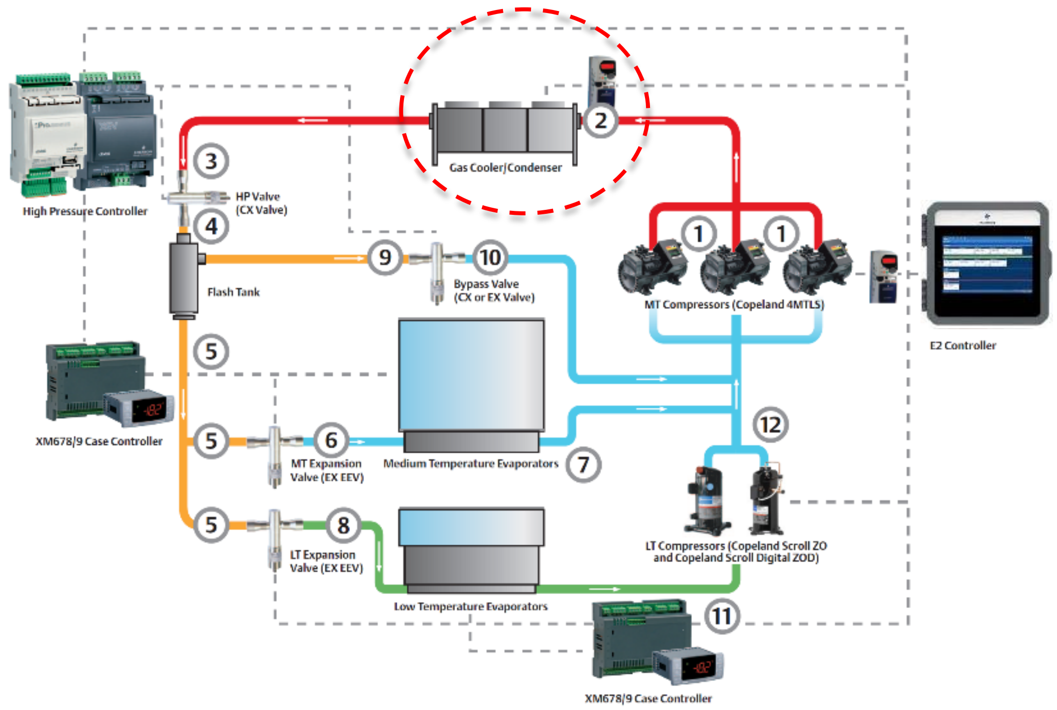
Transcritical Mode
(AKA Supercritical)

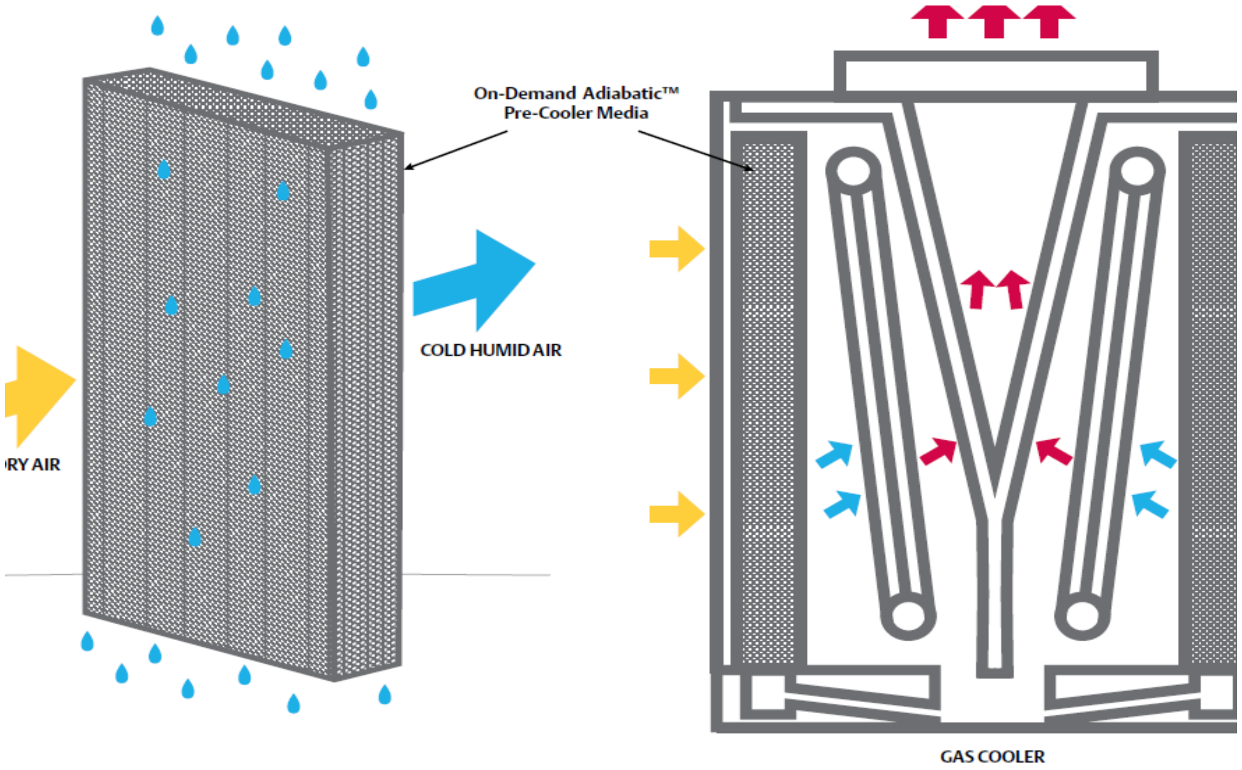
- > 75F Ambient
- **5 to 7°F TD**

Subcritical Mode

- <75°F Ambient
- **10 to 13°F TD**

CO_2 Transcritical Booster System Condenser / Gas Cooler

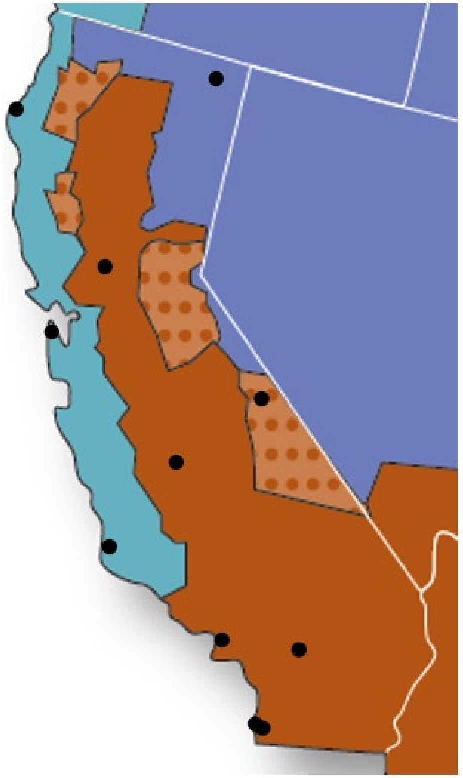




Zone	Hot Dry 2B	Max Temp	35F Bin	40F Bin	45F Bin	50F Bin	55F Bin	60F Bin	65F Bin	70F Bin	75F Bin	80F Bin	85F Bin	90F Bin	95F Bin	100F Bin	105F Bin	110F Bin	115F Bin	Total Hou
Non Adiabatic	PHOENIX SKY HARBOR INTL AP	111.92	14	127	345	560	710	787	739	722	746	735	780	831	671	526	350	117	0	8760
Adiabatic	PHOENIX SKY HARBOR INTL AP	76.88863	14	127	346	728	1621	1956	1885	1696	387	0	0	0	0	0	0	0	0	8760
			Subcritical operation								Transcritical operation									
			Percentage of total year spent at ambient BIN group																	
	Non Adiabatic	%	0.16	1.45	3.94	6.39	8.11	8.98	8.44	8.24	8.52	8.39	8.90	9.49	7.66	6.00	4.00	1.34	0.00	
	Adiabatic	%	0.16	1.45	3.95	8.31	18.50	22.33	21.52	19.36	4.42	0.00	0.00	0.00	0.00	0.00	0.00	0.00	0.00	
	Non Adiabatic	4004	%			65 up	6100	69.63	%		75-85	2261	25.81	%	90-105	2378	27.15			%
	Adiabatic	8373	%			65 up	3968	45.30	%		75-85	387	4.42	%	90-105	0	0.00			%

% Time Operating in Transcritical Mode
(aka Supercritical Operation)

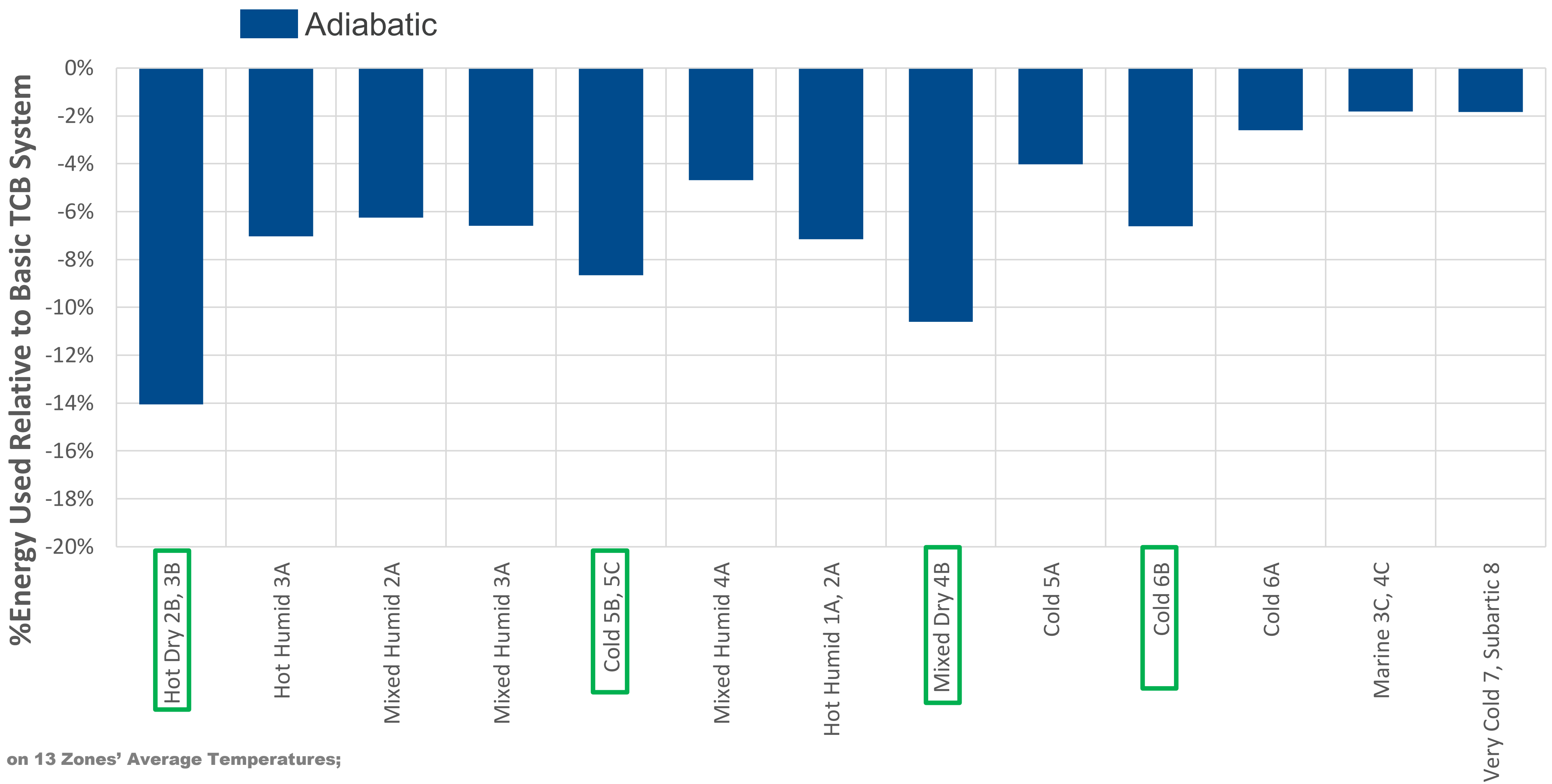
California
6 Zones



City	ASHRAE	IECC	%TC Dry GC	%TC Adi GC
San Diego	Hot Dry	3B	10%	0.5%
Sacramento	Hot Dry	3B	22%	0%
Los Angeles	Hot Dry	3B	5%	0%
Palm Springs	Hot Dry	3B	54%	6.5%
Fresno Yosemite	Hot Dry	3B	30%	0%
San Francisco	Marine	3C	2%	0%
Santa Maria	Marine	3C	4%	0%
Bishop	Mixed Dry	4B	25%	0%
Arcata	Marine	4C	0.1%	0%
Alturas	Cold	5B	13%	0%

Assumptions: ≥ 75 °F Ambient = Supercritical operation dry gas cooler
≥ 72 °F Ambient = Water flow adiabatic gas cooler

Percent of Energy Saving vs. Basic TCB Systems



Charts Based on 13 Zones' Average Temperatures;

Weather Data: NREL TMY3 data, EES Software, 400MBH MT +18SST, 100MBH LT -20F

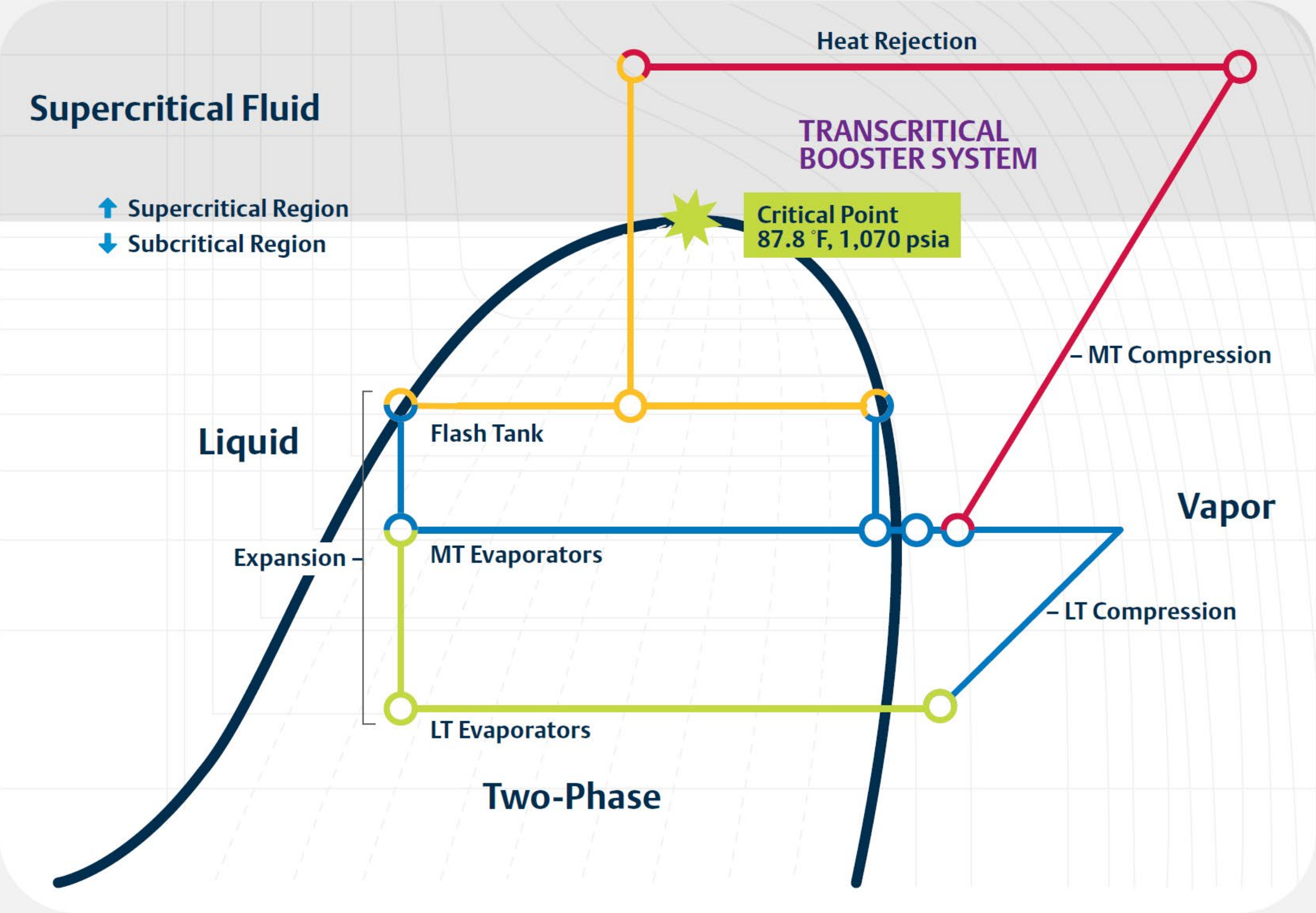
Climate Zones with Lower Average Relative Humidity Show Better Energy Reduction With Adiabatic Gas Coolers than with Parallel Compression…

3

HIGH OPERATING PRESSURES

CO_2 TCB system operating pressures are significantly higher than traditional HFC systems (i.e., those using R-404A or R-410A).

- Pressures can **exceed 1,400 psi** when MT compressors discharge into the gas cooler on a hot summer day (e.g., 100 °F).
- MT discharge lines must be constructed with stainless steel or special ferrous alloy copper to handle these pressures.



High Pressure Safety Mitigation

MT Compressor are rated for Minimum 1740 psig operating

High Pressure rated Piping For Discharge & other Sections

Pressure-test with dry nitrogen and evacuate.

Verify if pressure-relief valves are installed in accordance with local building codes.

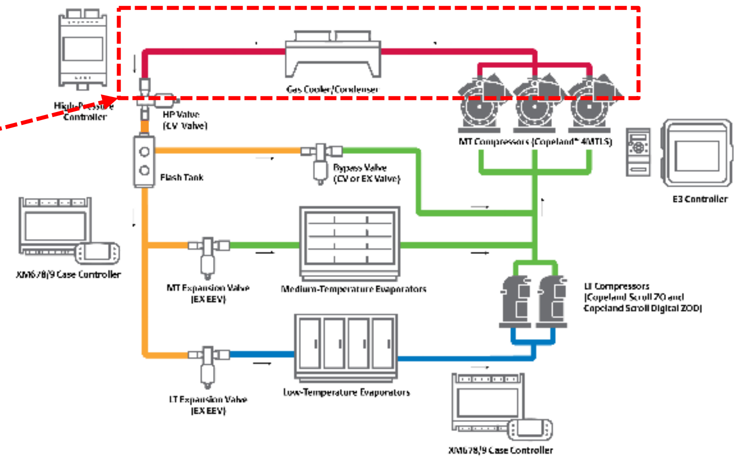
Must read all manuals/instructions provided by manufacturers and consult all applicable safety materials.

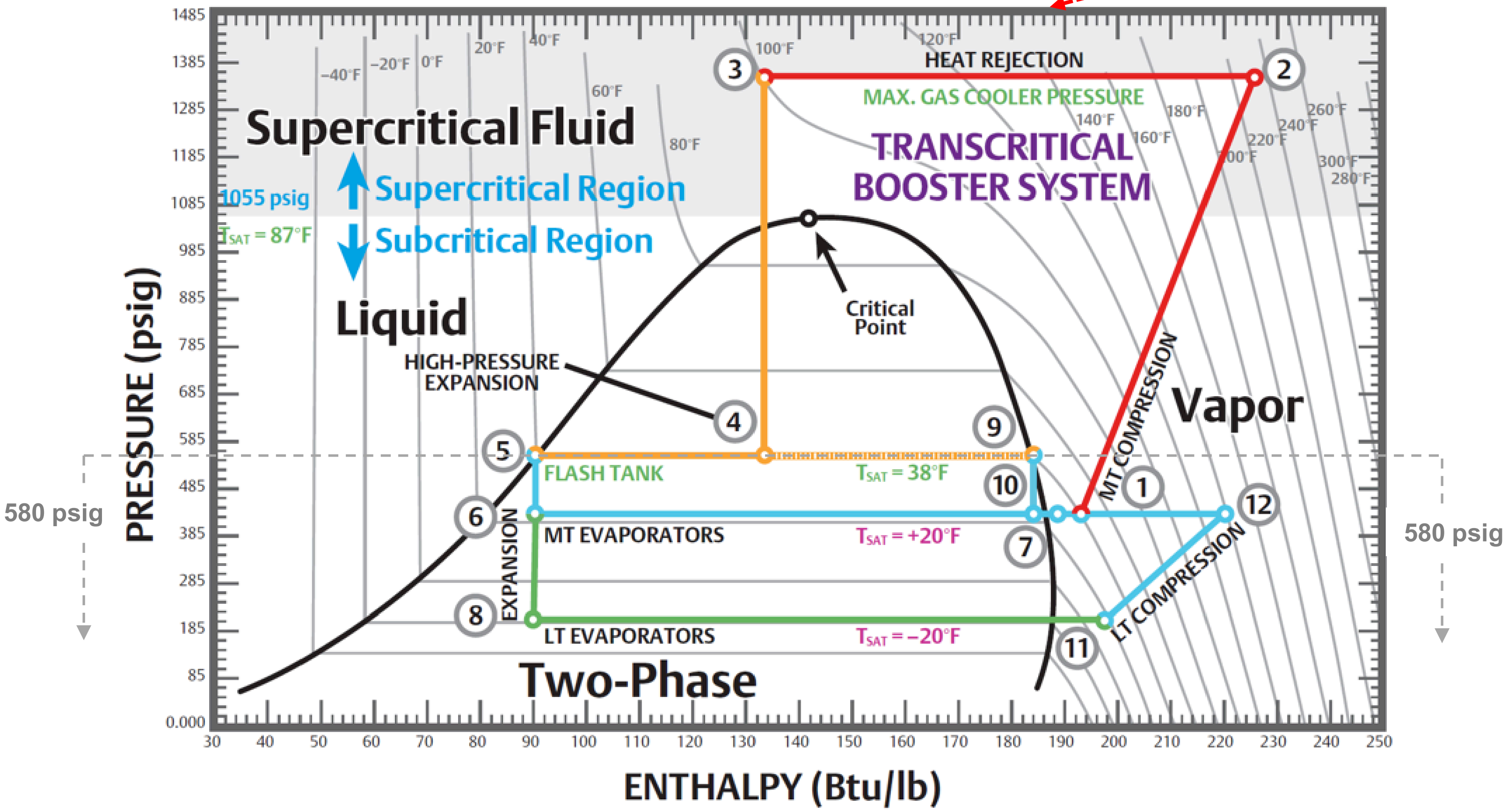
Differences: R-744 vs. HFC

High Operating Pressures

- Safety awareness

Where They Occur





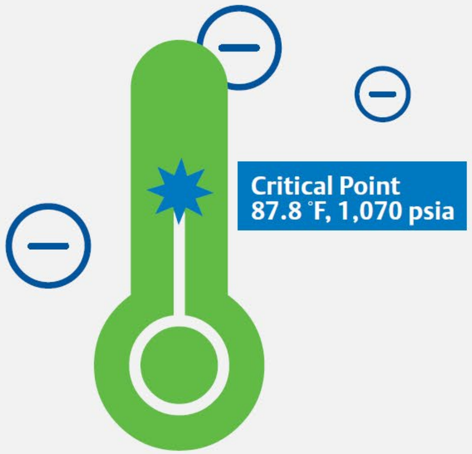
4

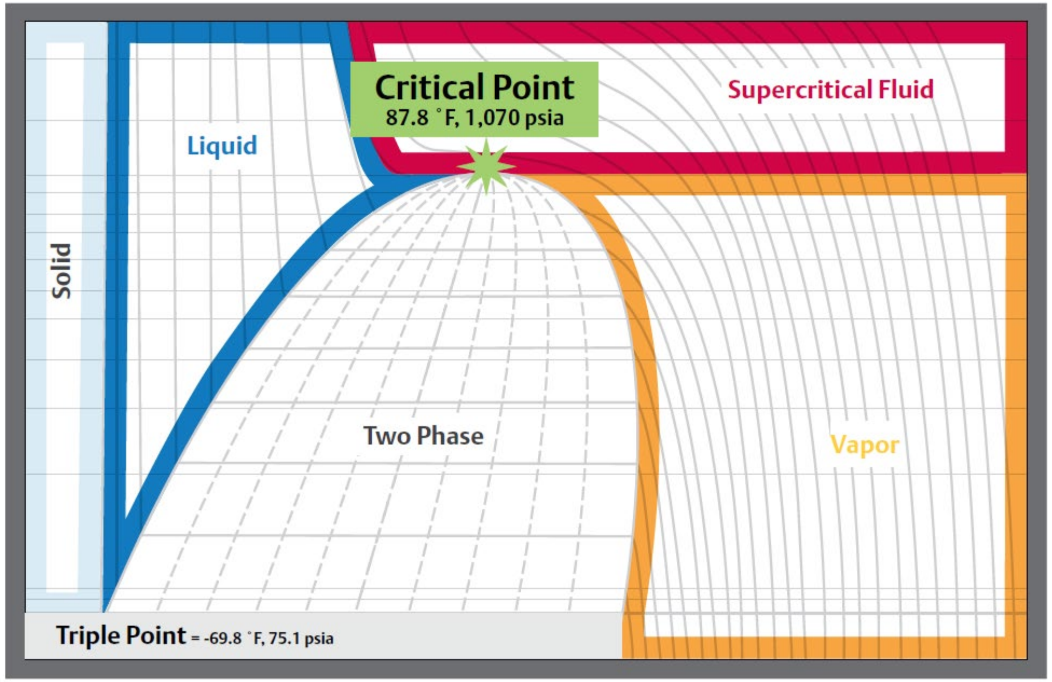
LOW CRITICAL POINT OF 87.8 °F

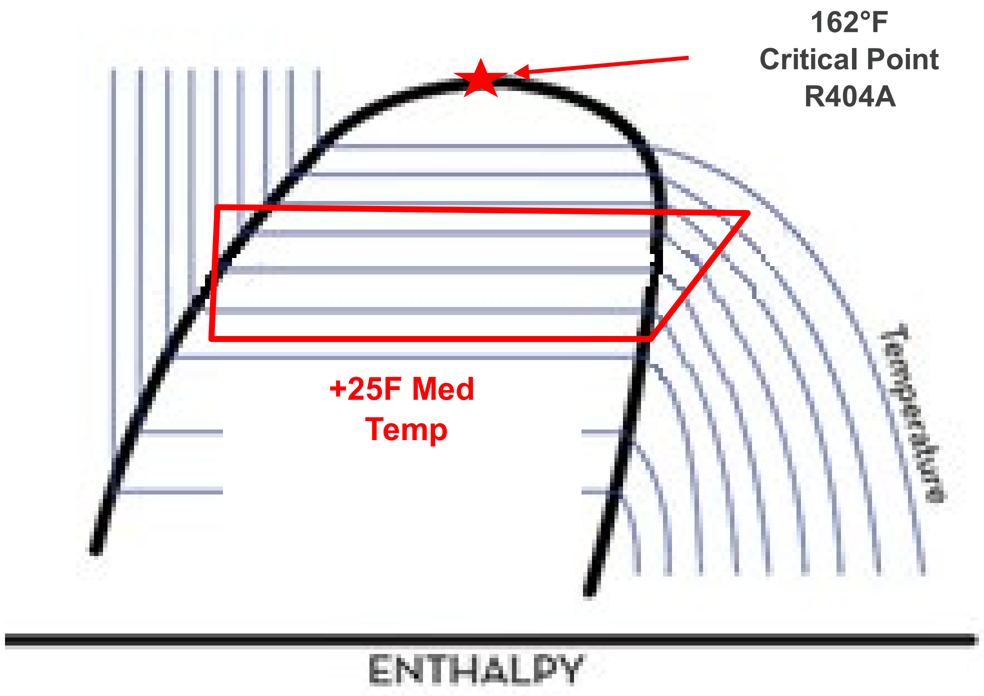
R-744 has a critical point of **87.8 °F**, which is relatively low from a refrigeration perspective.

When the ambient temperature rises above approximately **75 °F**, system conditions cause the refrigerant to exit the gas cooler as a *supercritical* fluid (at or above **87.8 °F**) and run in *transcritical* mode, where its pressure and temperature relationships can rise and fall independently.

R-744 is at saturation when liquid and vapor coexist below the critical point. When the system operates below 87.8 °F, it is referred to as *subcritical* mode.





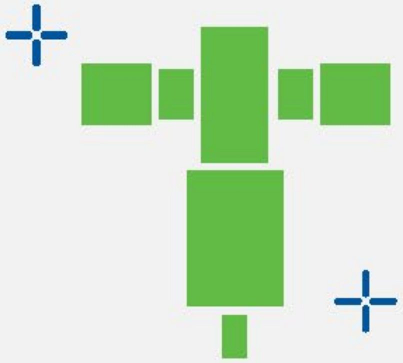


5

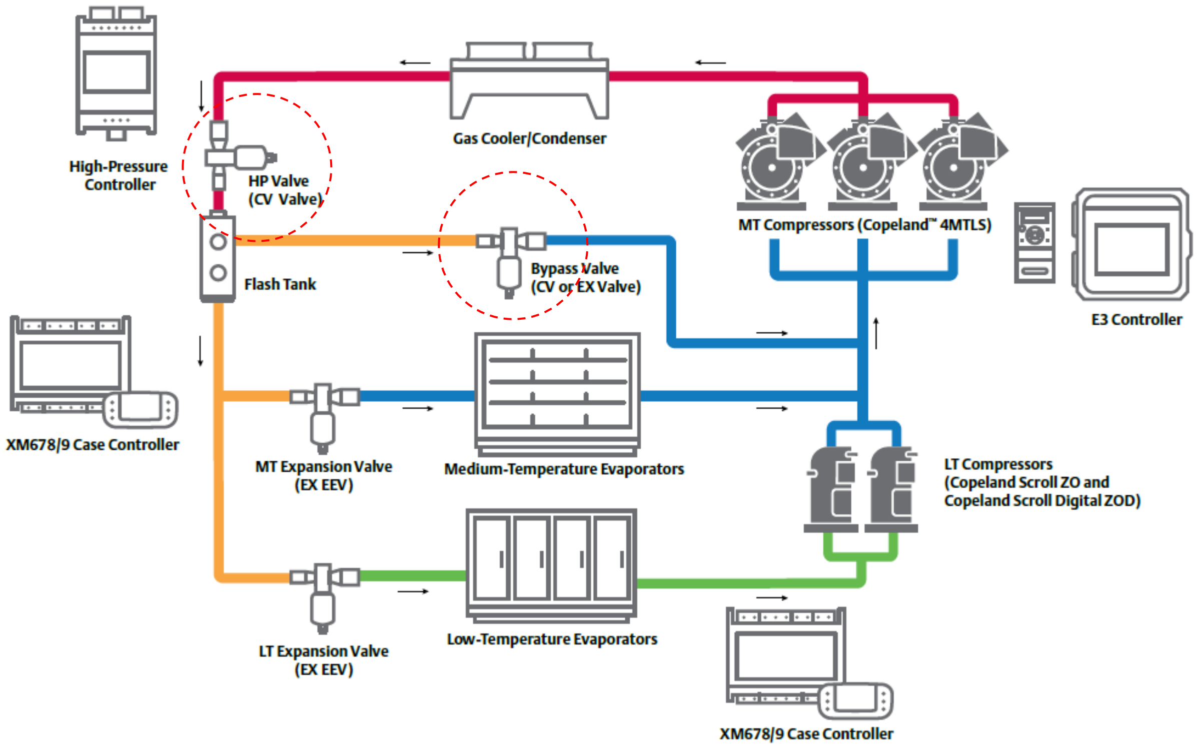
HIGH-PRESSURE VALVE & BYPASS VALVE

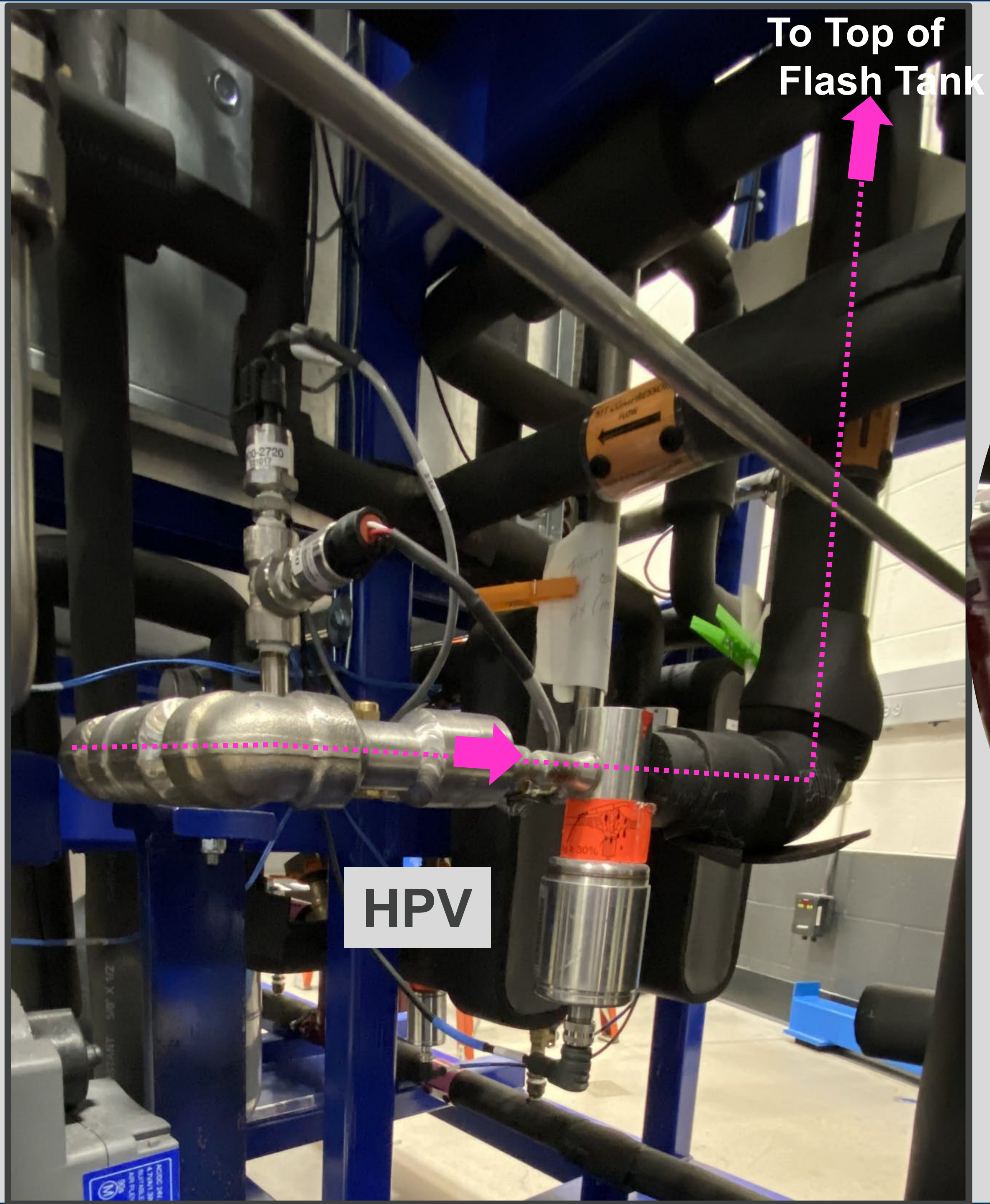
As R-744 exits the gas cooler, its pressures must be reduced before being reintroduced to the refrigeration circuits.

A high-pressure valve (HPV) reduces the refrigerant to around **550 psi** and transfers it to a receiver (aka flash tank) that separates vapor from liquid



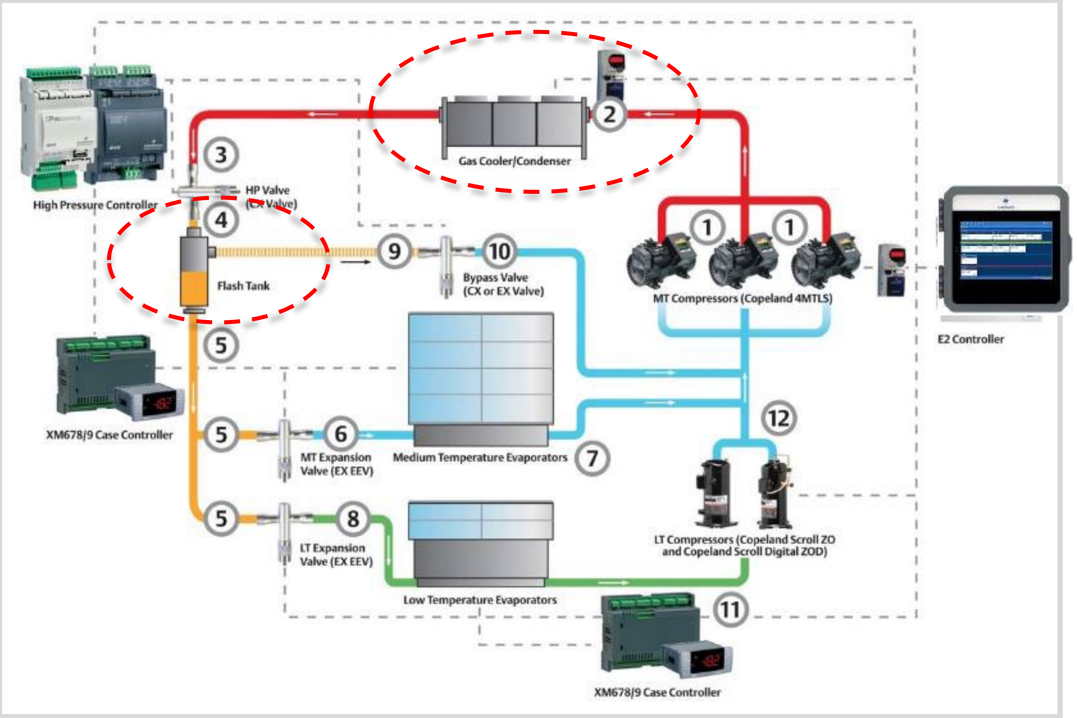
HIGH-PRESSURE VALVE



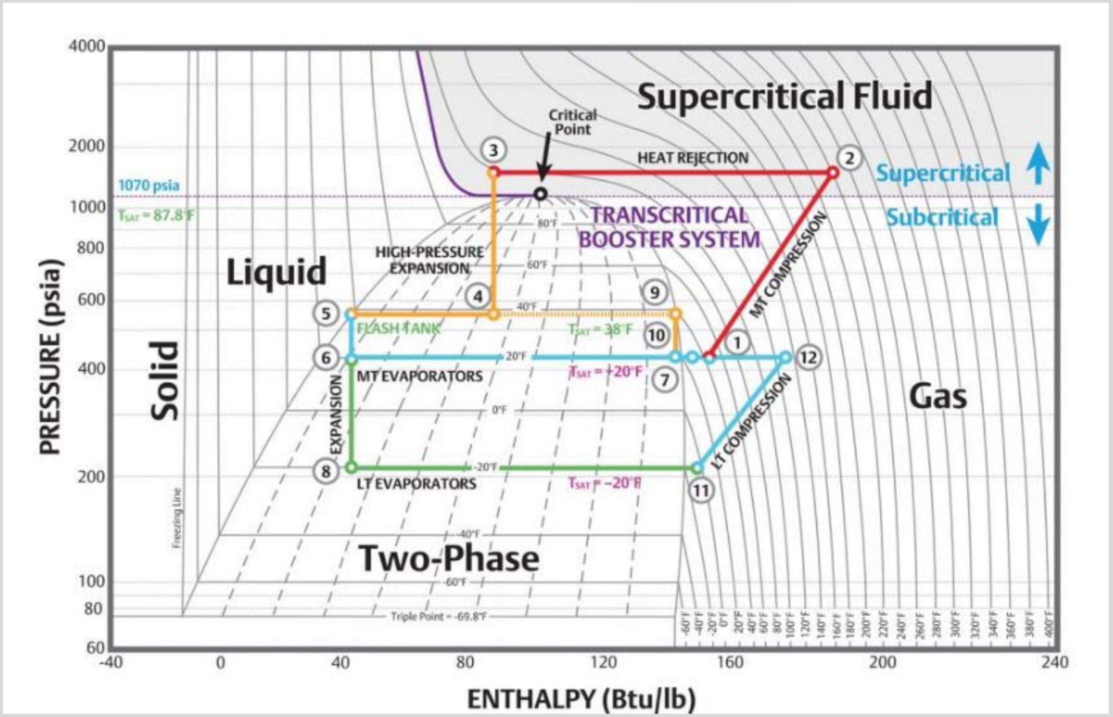

To Top of
Flash Tank
HPV


BGV

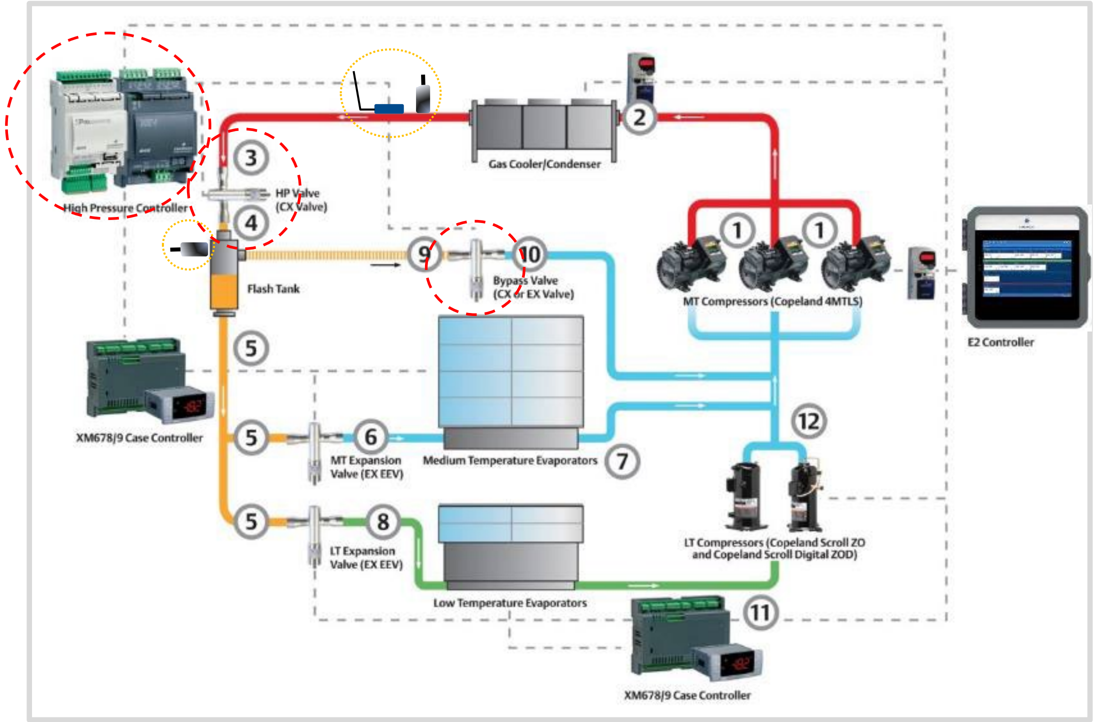
CO_2 Booster Refrigeration System
Gas Cooler / Condenser & Receiver / Flash Tank



- Condenses below Critical Temp
- Cools the Discharge Gas Above Critical Temp
- Flash Tank Separates Gas from Liquid
- Liquid is pulled off the bottom to feed MT and LT cases.



CO_2 High Pressure Controller



Inputs
1. Gas Cooler Out Pressure
2. Gas Cooler Out Temp.
3. Flash Tank Pressure
4. Capacity Demand Input

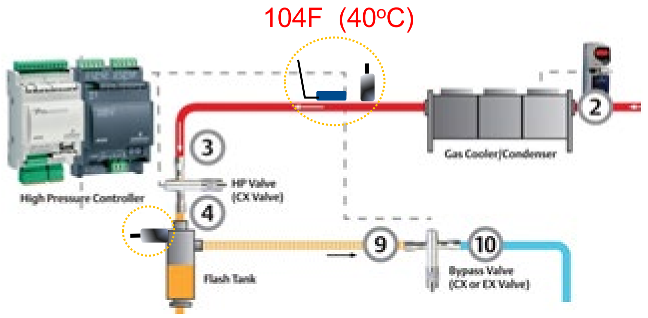
Subcritical Operation
- Maintains Subcooling In Condenser

Transcritical Operation
- Ignores Subcooling Control & Controls Gas Cooler Pressure

Transient Operation
- Avoids Hard Switch In Either Sub or Transcritical To Evade Effects of Rapid CO2 Density Change

High Pressure Valve (HPV) & Bypass Valve (BPV)
–The Control Point In Both The Valves Is Flash Tank Pressure
– If Pressure Is > Set Point, The HPV Throttle & BPV Opens
– If Pressure Is < Set Point. The HPV Opens & BPV Throttles

Three Examples Same Evaporator Conditions



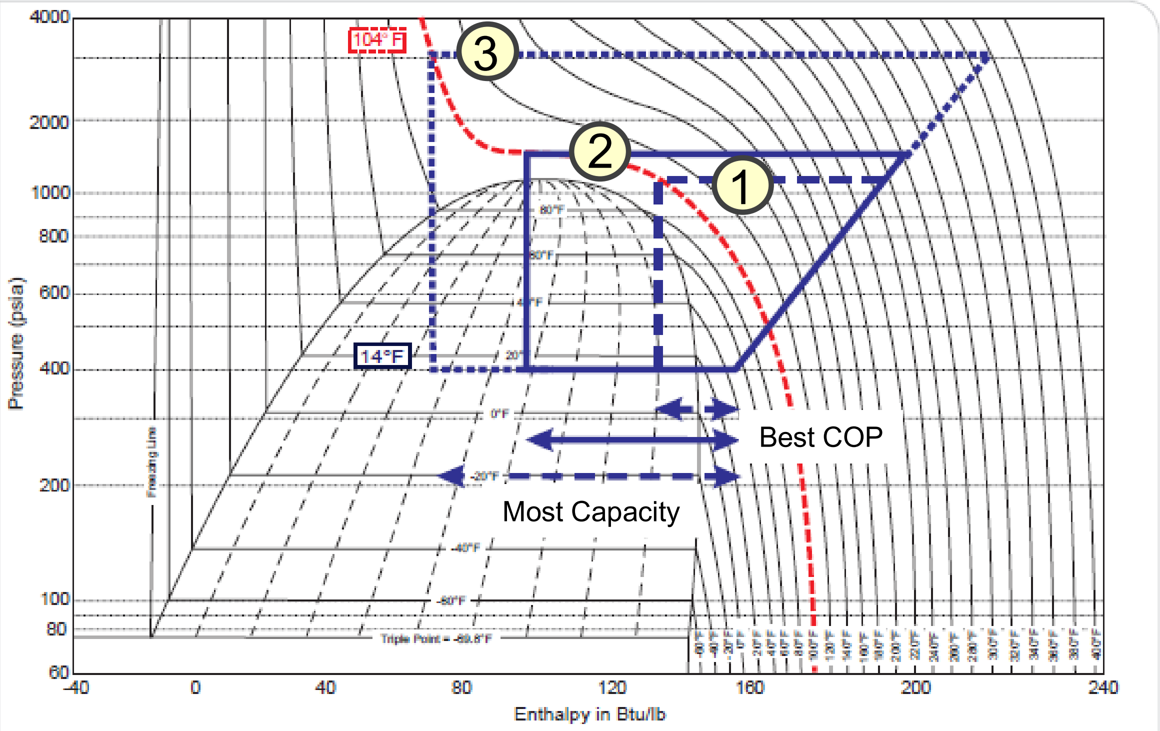
COP= <u>Heat Energy Removed (BTU)</u>
Power Input

Optimal GC Pressure
In TC Mode Only

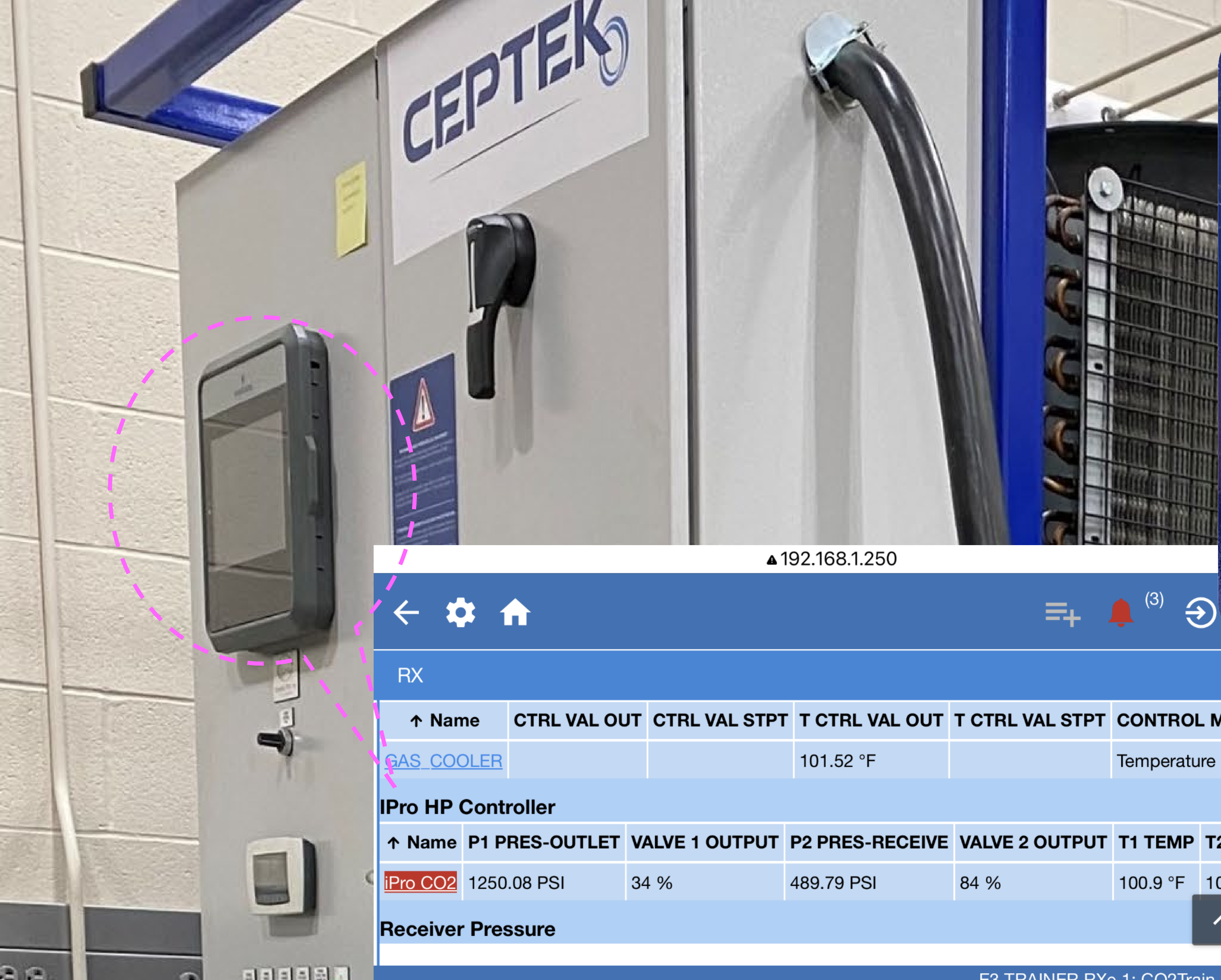
2

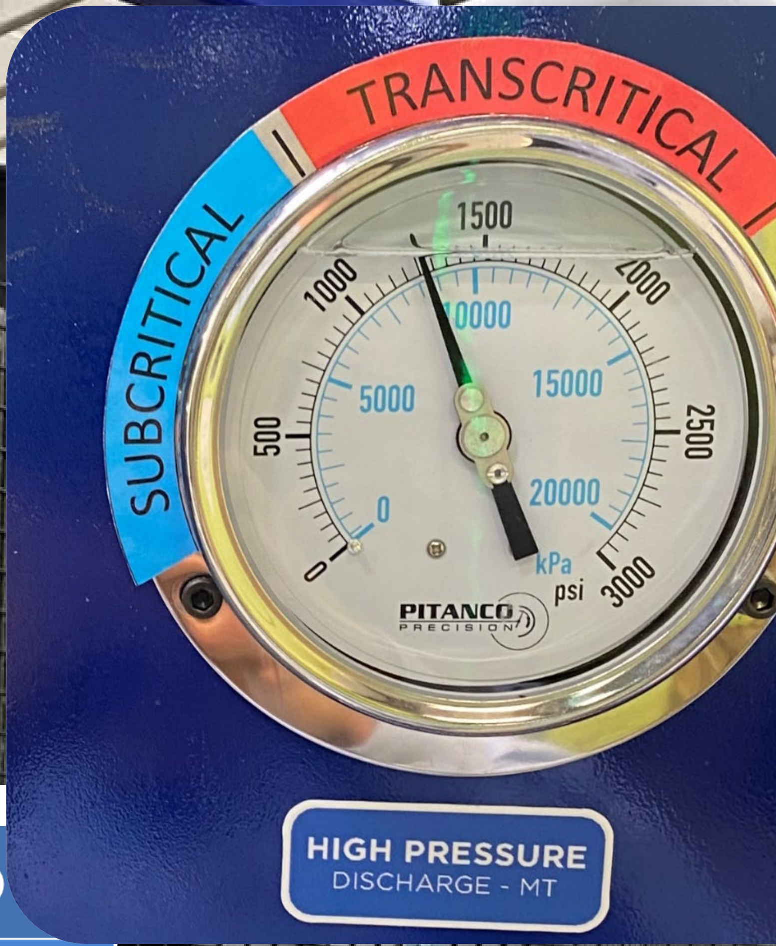
	30	1088	75
	31	1111	76.6
	32	1147	79.1
	33	1182	81.5
	34	1218	84
	35	1253	86.4
	36	1288	88.8
	37	1324	91.3
	38	1359	93.7
	39	1400	96.5
	40	1430	98.6
	41	1465	101
	42	1501	103.5

2: Optimal pressure for gas cooler outlet temperature



For each example , R744 exits the gas cooler at 104ºF. This exit temperature is a function of the size of the gas cooler and the ambient temperature, in the same way as condensing temperature is a function of the size of the condenser and the ambient temperature.





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(3)

RX

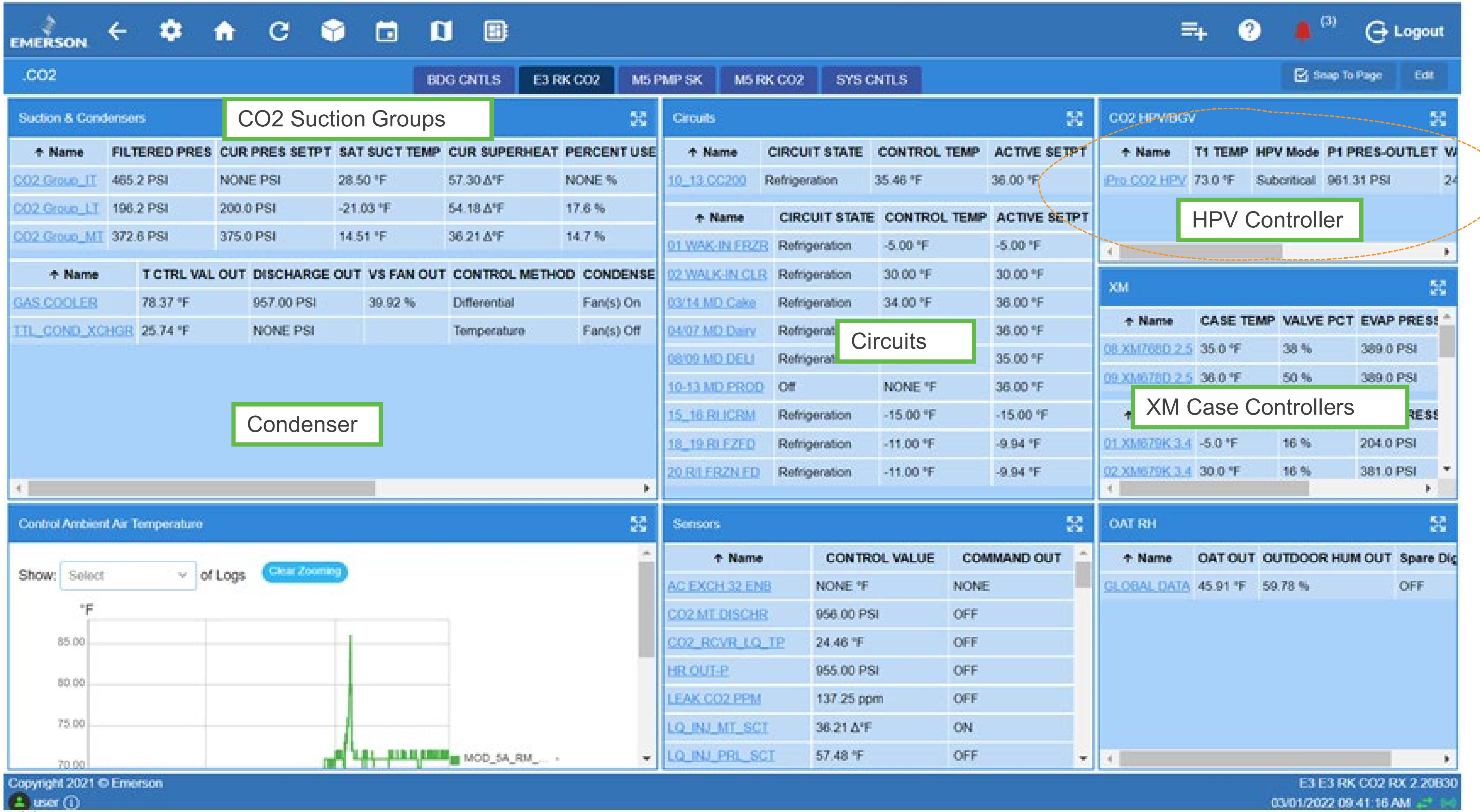
↑ Name	CTRL VAL OUT	CTRL VAL STPT	T CTRL VAL OUT	T CTRL VAL STPT	CONTROL METHOD
GAS_COOLER			101.52 °F		Temperature

IPro HP Controller

↑ Name	P1 PRES-OUTLET	VALVE 1 OUTPUT	P2 PRES-RECEIVE	VALVE 2 OUTPUT	T1 TEMP	T2 TEMP
iPro CO2	1250.08 PSI	34 %	489.79 PSI	84 %	100.9 °F	100.9 °F

Receiver Pressure

Lumity Software User Interface Can Be Customized To User Preference

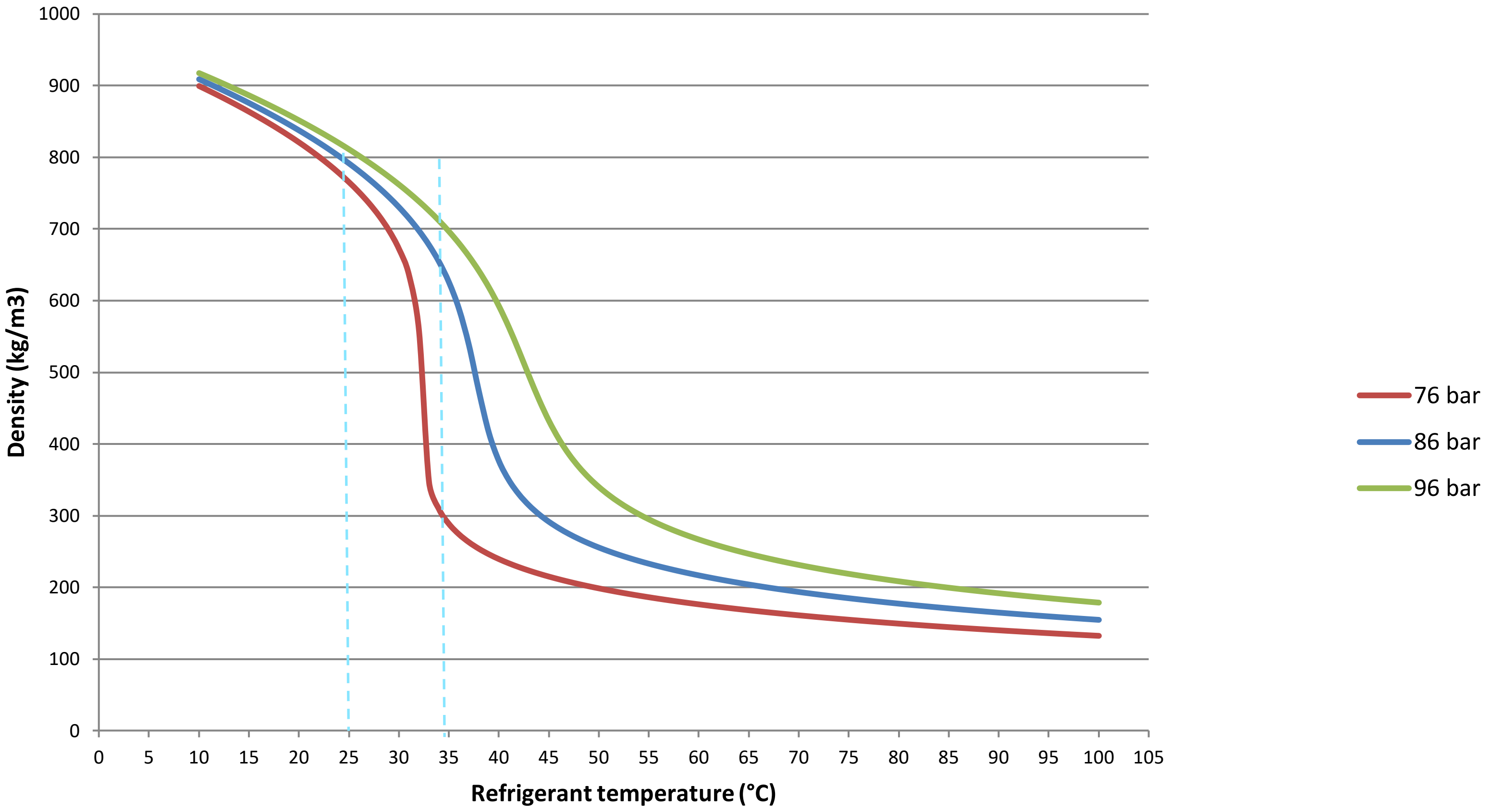


Lumity Software User Interface Can Be Customized To User Preference

T1 TEMP	75.0 °F
HPV Mode	Subcritical
SETPOINT	5.00
CONTROL VALUE	7.0
P1 PRES-OUTLET	983.50 PSI
VALVE 1 OUTPUT	20 %
BGV SETPOINT	440.00 PSI
P2 PRES-RECEIVE	440.91 PSI
VALVE 2 OUTPUT	23 %
ENABLE	ON

Great Screen For Troubleshooting

CO2 Density (kg/m3)



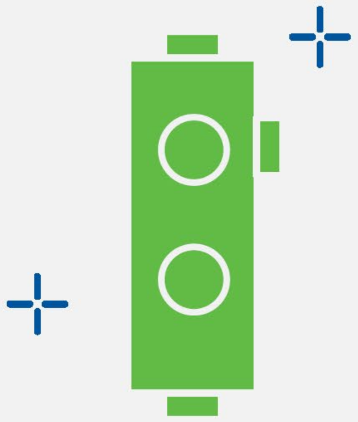
The design of the gas cooler pressure regulating valve is very sensitive, as the refrigerant density changes rapidly between 25 and 35°C.(77F & 95F) The proper selection of the expansion valve therefore requires checking different operating points.

6

FLASH TANK AND/OR RECEIVER

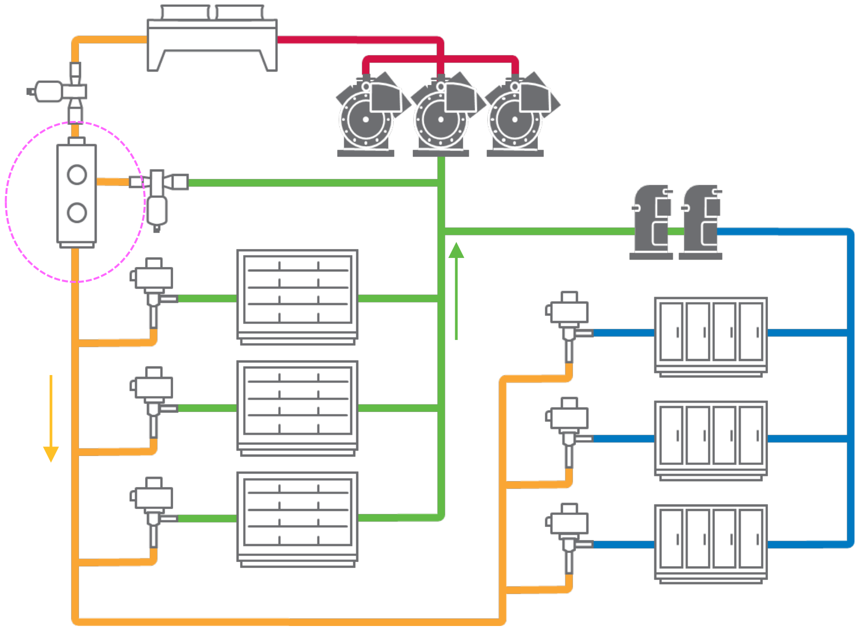
The flash tank receives a mixture of vapor and liquid refrigerant at around **40 °F** equivalent saturation, with vapor rising to the top and liquid settling at the bottom.

- Liquid is circulated through insulated lines that feed the MT and LT cases (as low as **-20 °F**) — all of which are equipped with an electronic expansion valve (EEV).
- To help relieve flash tank pressures, vapor is fed to a bypass line through a bypass gas valve (BGV) located at the upper right of the tank.



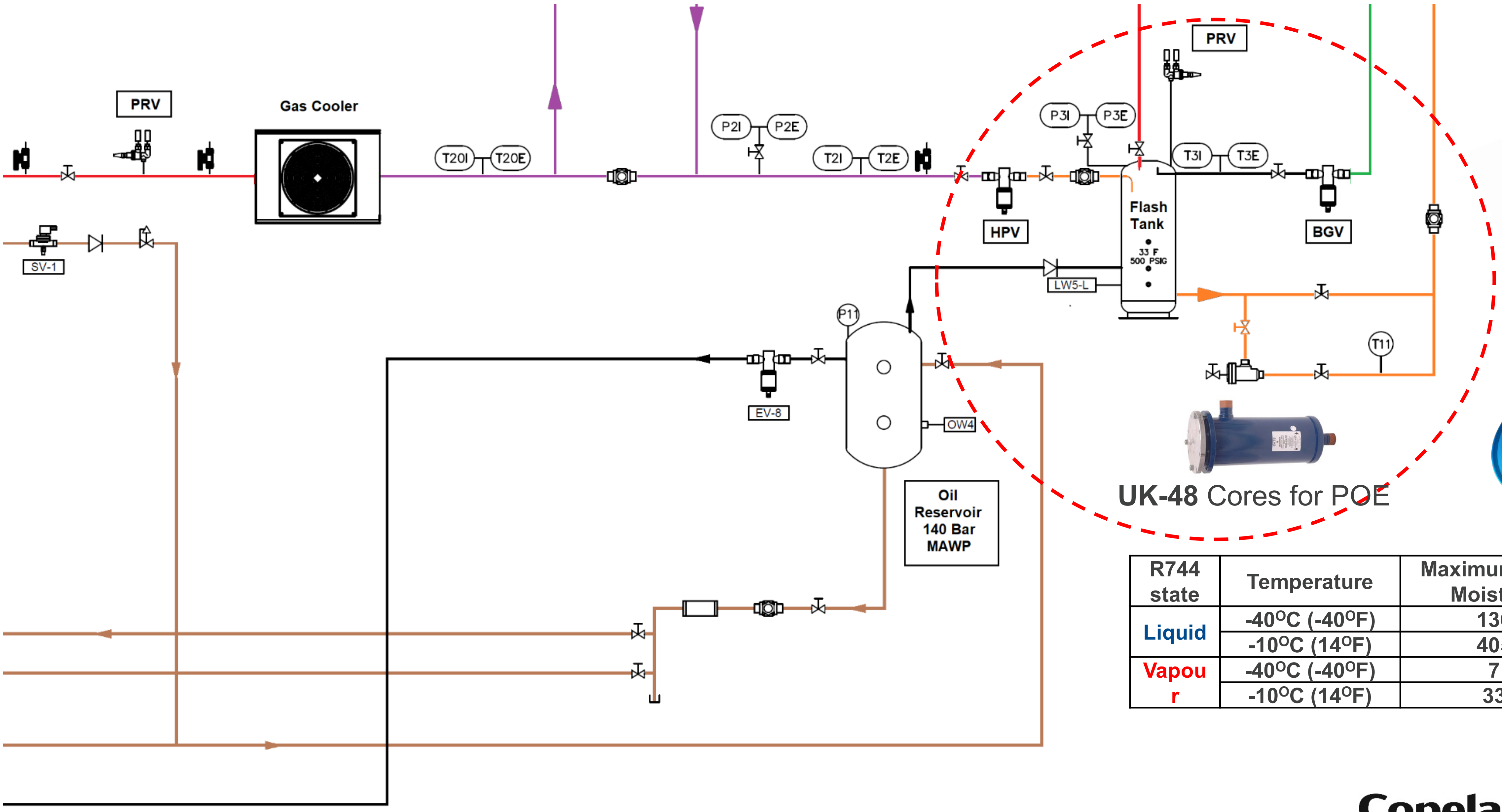
FLASH/RECEIVER TANK

- Insulated
- 38F = 537 pisg
- Sizing is Key
- Level Management



Stable Flash Tank pressure is critical to smooth performance Year round

Flash Tank



R744 state	Temperature	Maximum ppm Moisture
Liquid	-40°C (-40°F)	130
	-10°C (14°F)	405
Vapour	-40°C (-40°F)	7
	-10°C (14°F)	33

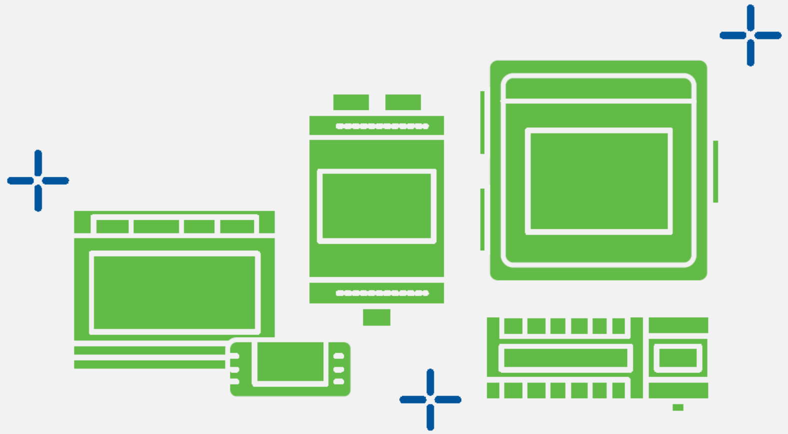
Copeland™ brand products

7

RELIANCE ON ELECTRONICS

Because R-744 is a very dynamic refrigerant that reacts quickly to changes in pressures and temperatures, electronic controls are required to perform a variety of key system functions:

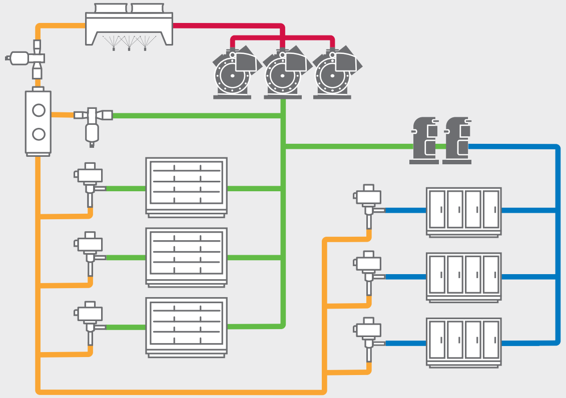
- Managing system pressures
- Controlling variable fan speeds
- Modulating the HPV, BGV and EEVs
- Maintaining a consistent flash tank pressure
- Assuring smooth compressor staging



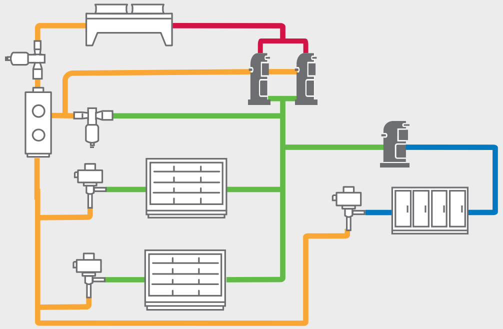
ELECTRONIC CONTROLLERS

CO2 System Solutions for Commercial & Industrial Refrigeration

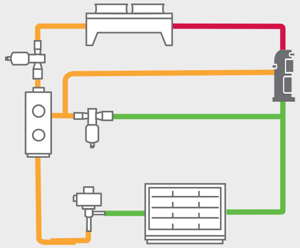
Centralized CO_2 Transcritical Booster



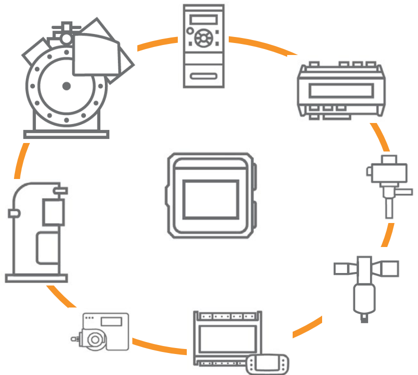
Distributed CO_2 Transcritical Booster

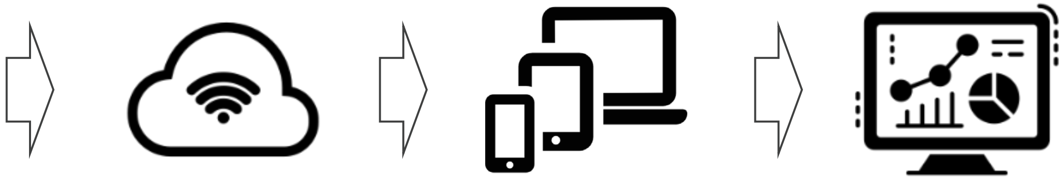


CO_2 Condensing Unit
CO_2 Heat Pumps



Vilter Single Screw



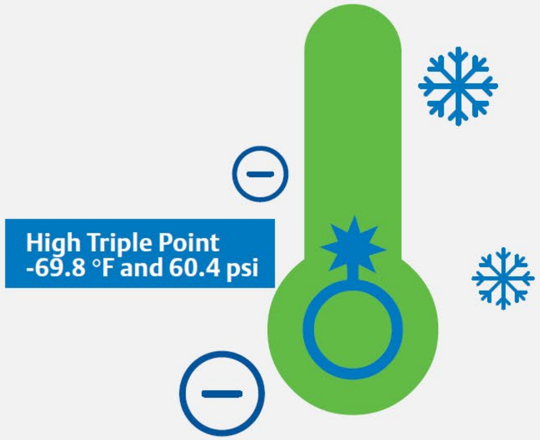


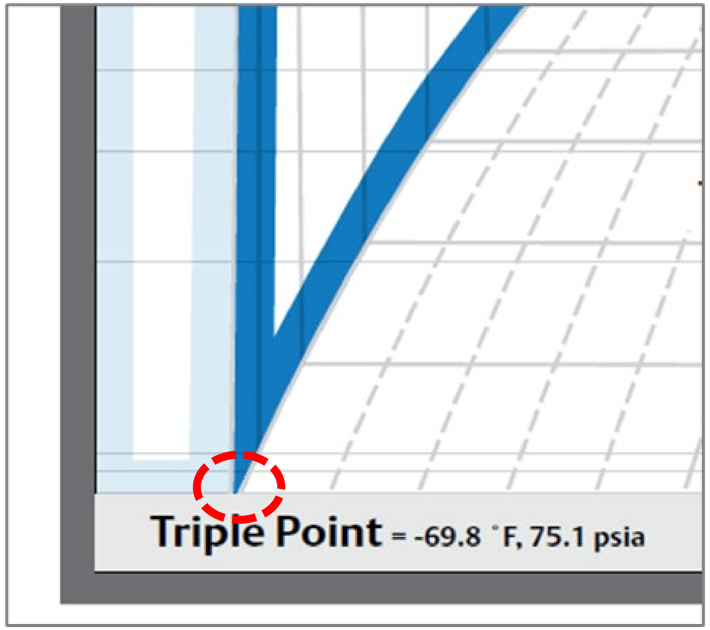
8

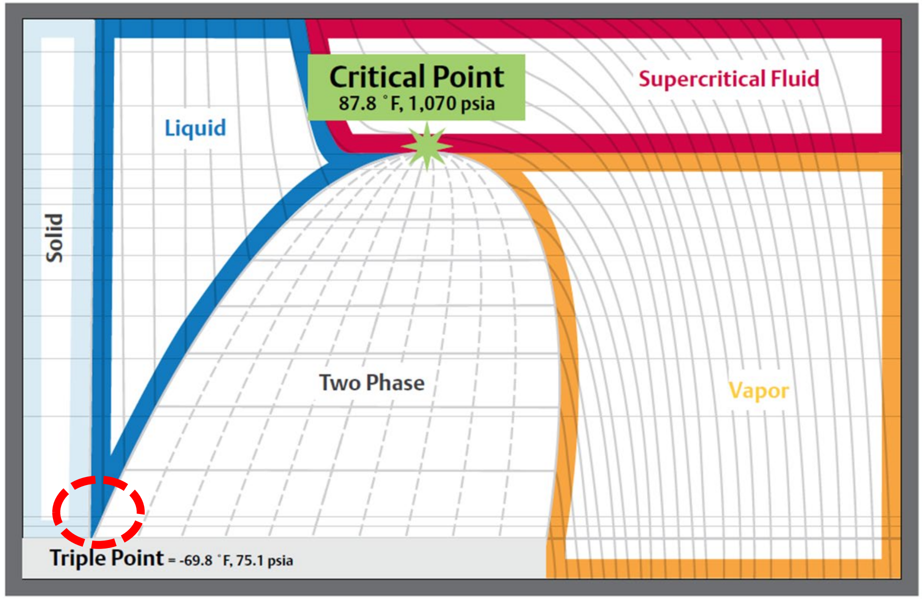
HIGH TRIPLE POINT OF -69.8 °F AND 60.4 PSI

-69.8 °F is well below normal operating ranges, but R-744's corresponding saturation pressure of **60.4 psi** can occur, especially during system charging.

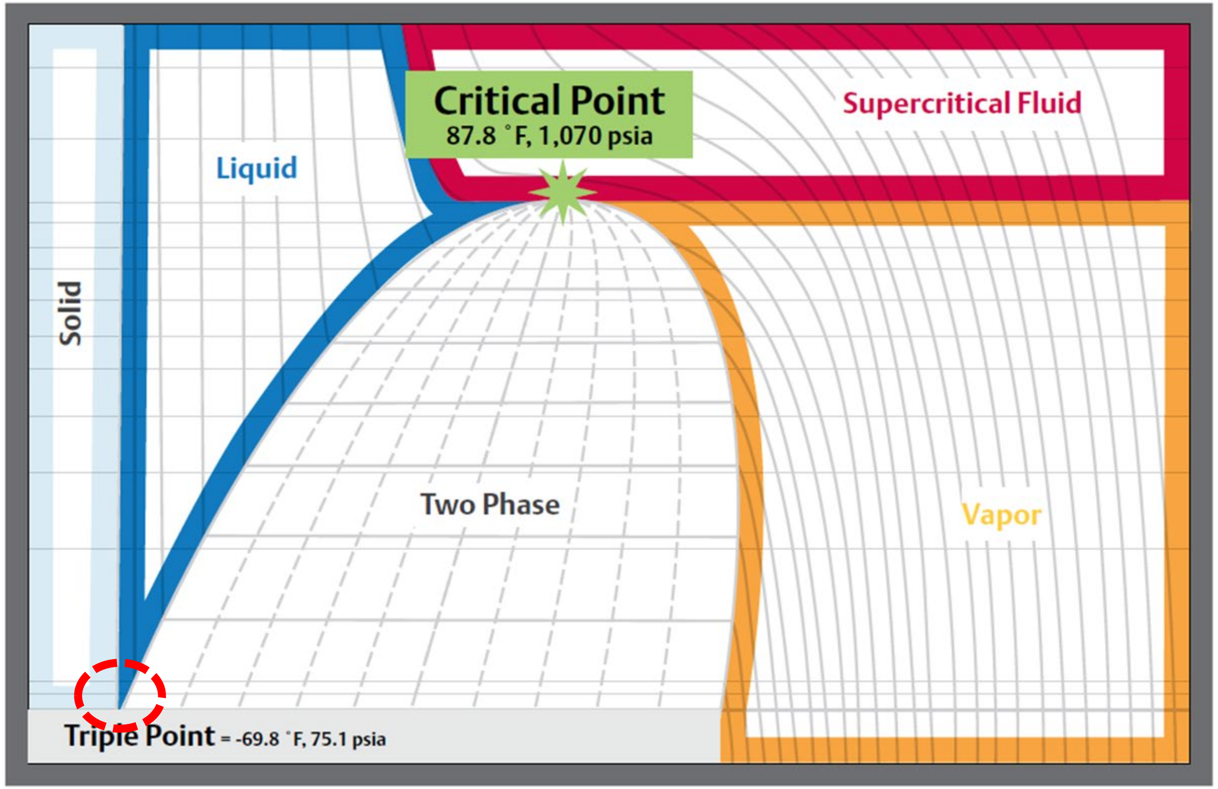
- When charging with liquid while system pressures are below the triple point, R-744 turns into dry ice, stops the refrigerant flow, and causes a variety of potential system problems.
- Technicians should charge with vapor until the system reaches **100 psi**, which is safely above R-744's triple point pressure.

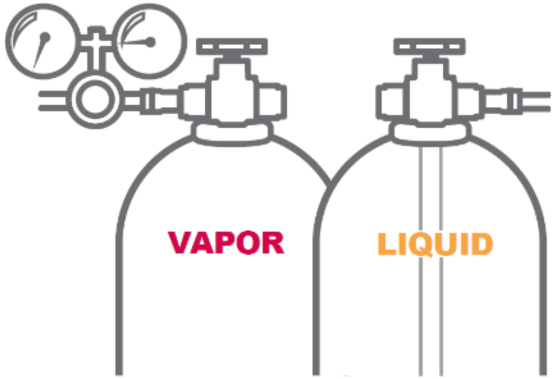






High Triple Point (60.4 psig, -69.8 °F)





-109.3 °F =
Surface Temp. of Dry Ice

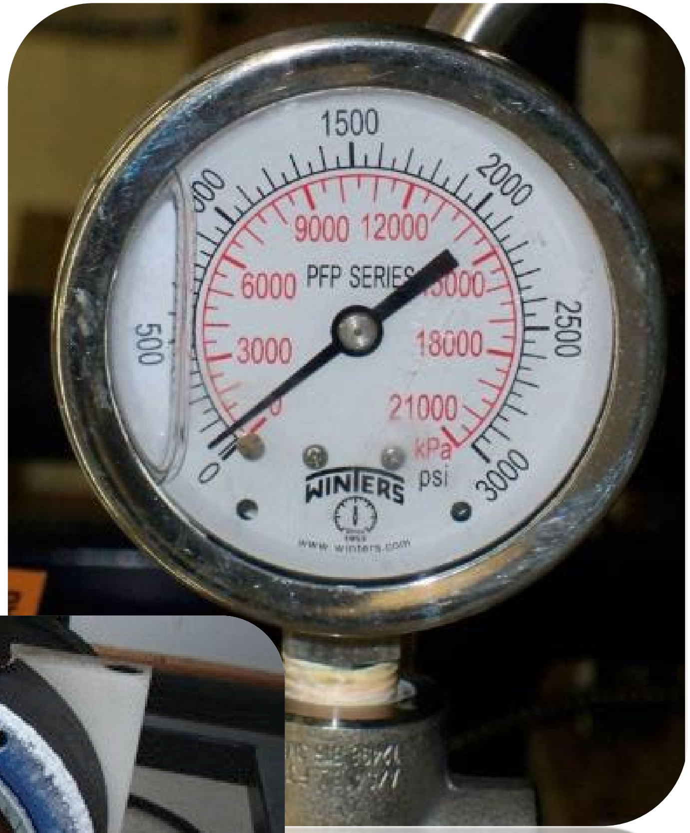
- Dry ice formation could form a blockage in a charging line
- Pressure behind the blockage will quickly rise as the dry ice sublimes
- This blockage or “plug” will shrink in size & High pressure behind the blockage could blow the plug out

Charging

- Break vacuum with CO_2 vapour
- Refrigerant to be **first charged as a vapor**
- Charge vapour into the system **high side**
- Charge refrigerant vapor **slowly**
- Keep vapour pressure above the triple point to prevent formation of dry ice

Triple Point = 61 psig (-70 °F)

(4.2 bar g) (-56 °C)

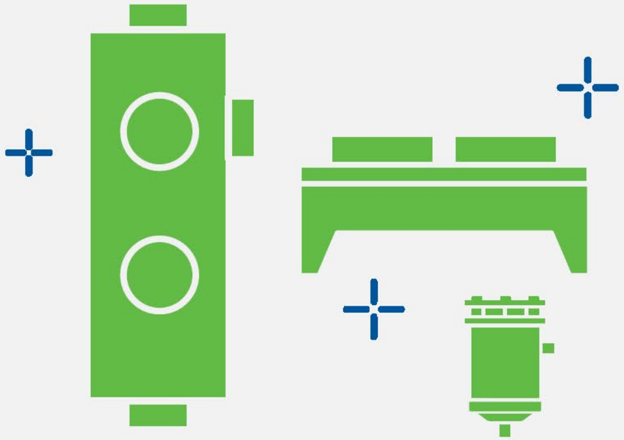


9

STANDSTILL PRESSURES

Managing the potential for rising system pressures during power outages or planned shutdowns is another important CO_2 TCB system design consideration.

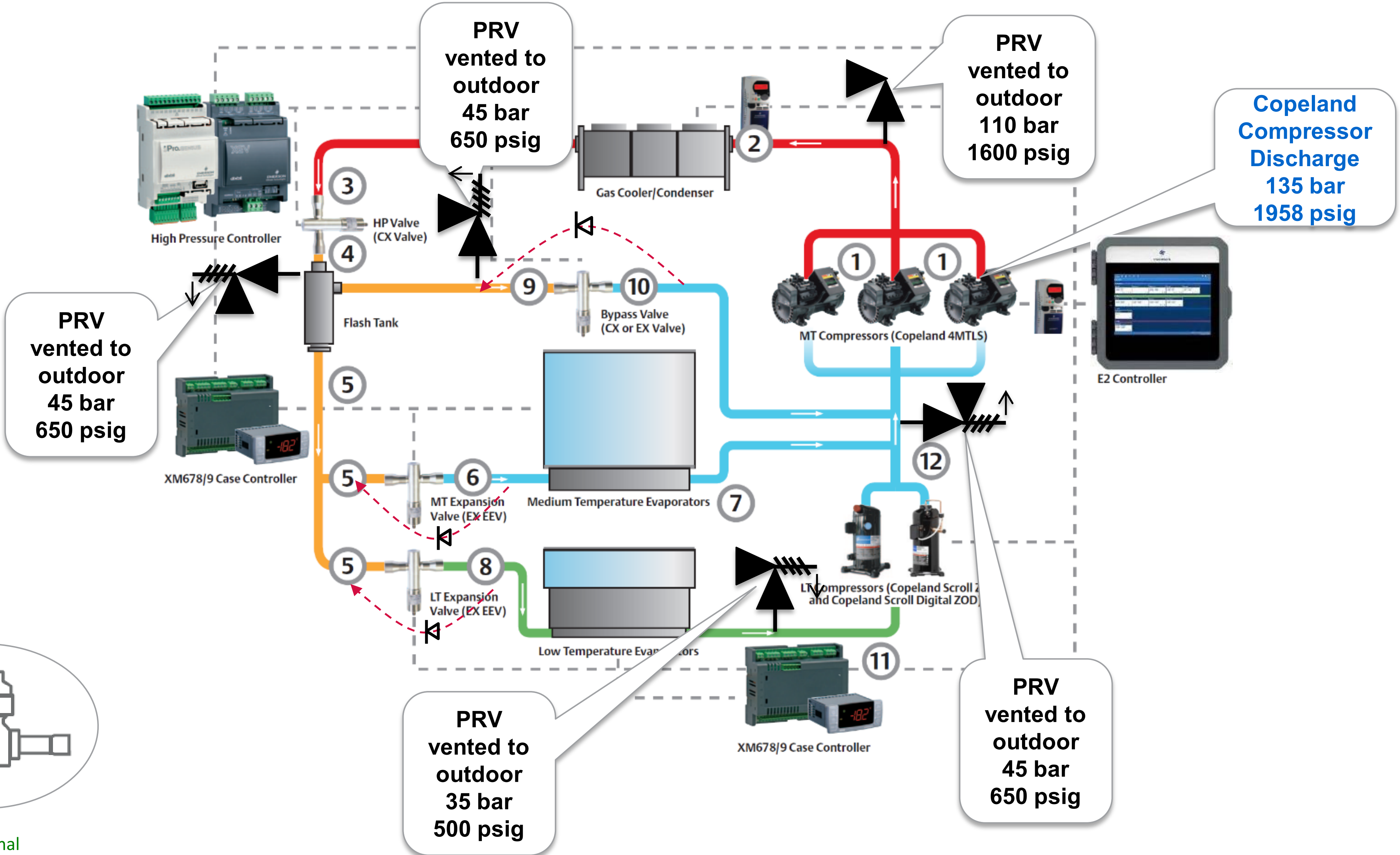
- Designers must understand the maximum pressure ratings of cases, valves, evaporators and all system components.
- A system should be designed to maintain the integrity of its weakest point and/or the component with the lowest safe working pressure.



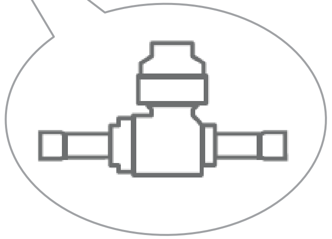
- 10 °C → 44 bar g (50 °F → 638 Psig)
- 20 °C → 56 bar g (68 °F → 815 Psig)
- 30 °C → 71 bar g (86 °F → 1032 Psig)

CO_2 Booster Refrigeration System

Understanding Pressure Reliefs – Will Vary based on OEM & Region



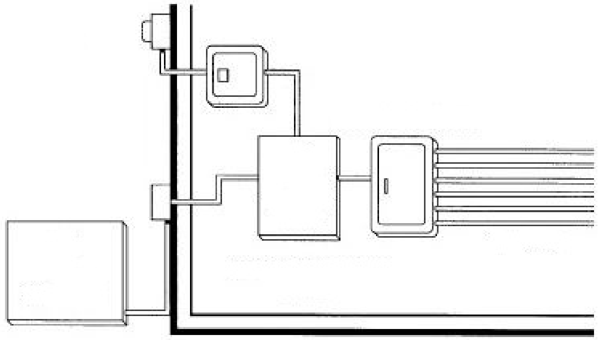
Did you leave it open or closed when servicing?



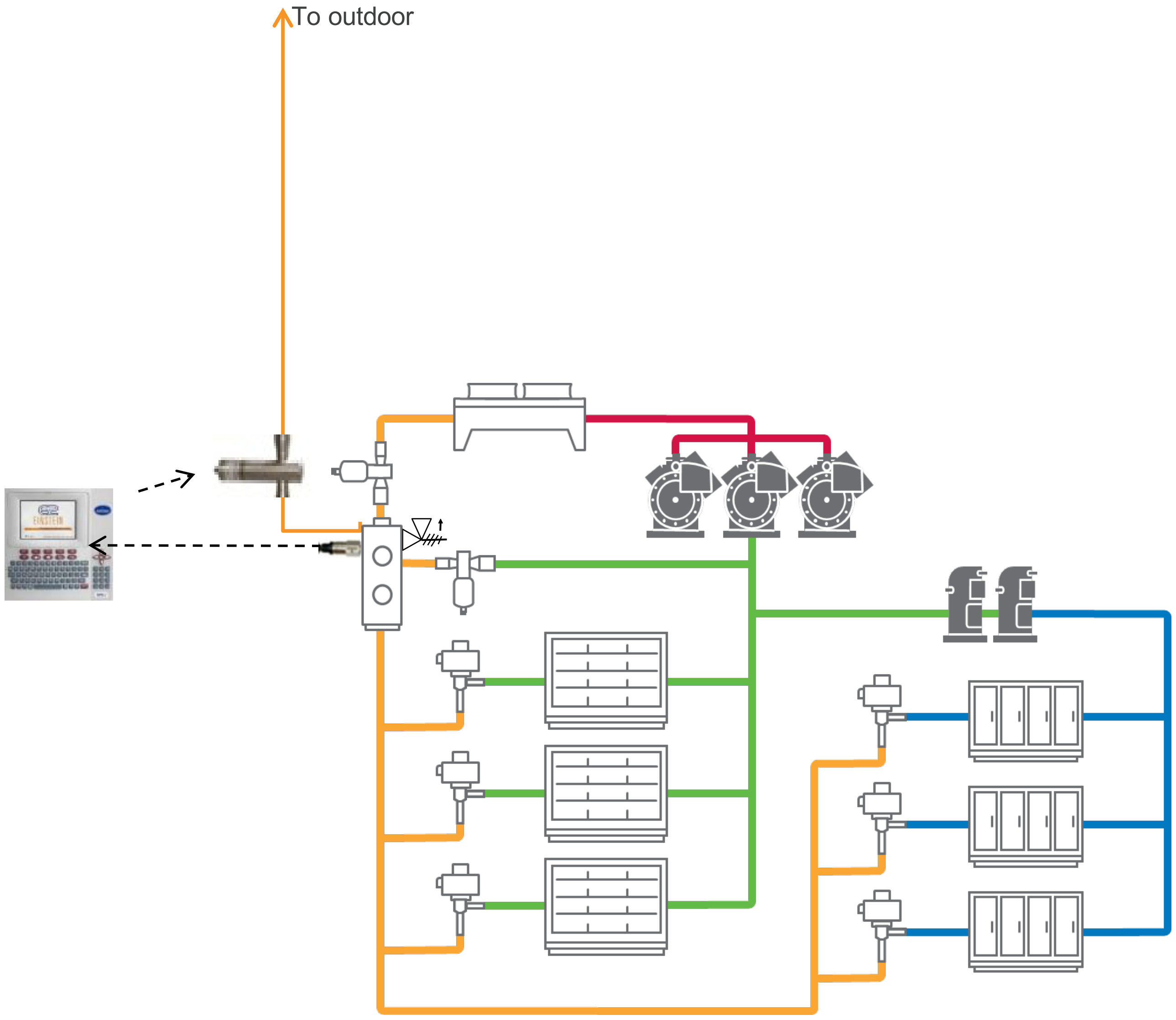
Separate Pressure Relief System

Advantage;

- ✓ Less stress on PRV
- ✓ Less CO_2 discharged Vs PRV
- ✓ Lowers overall maintenance costs

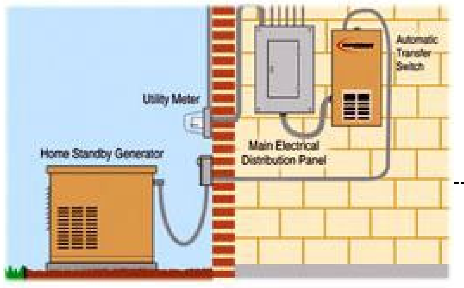


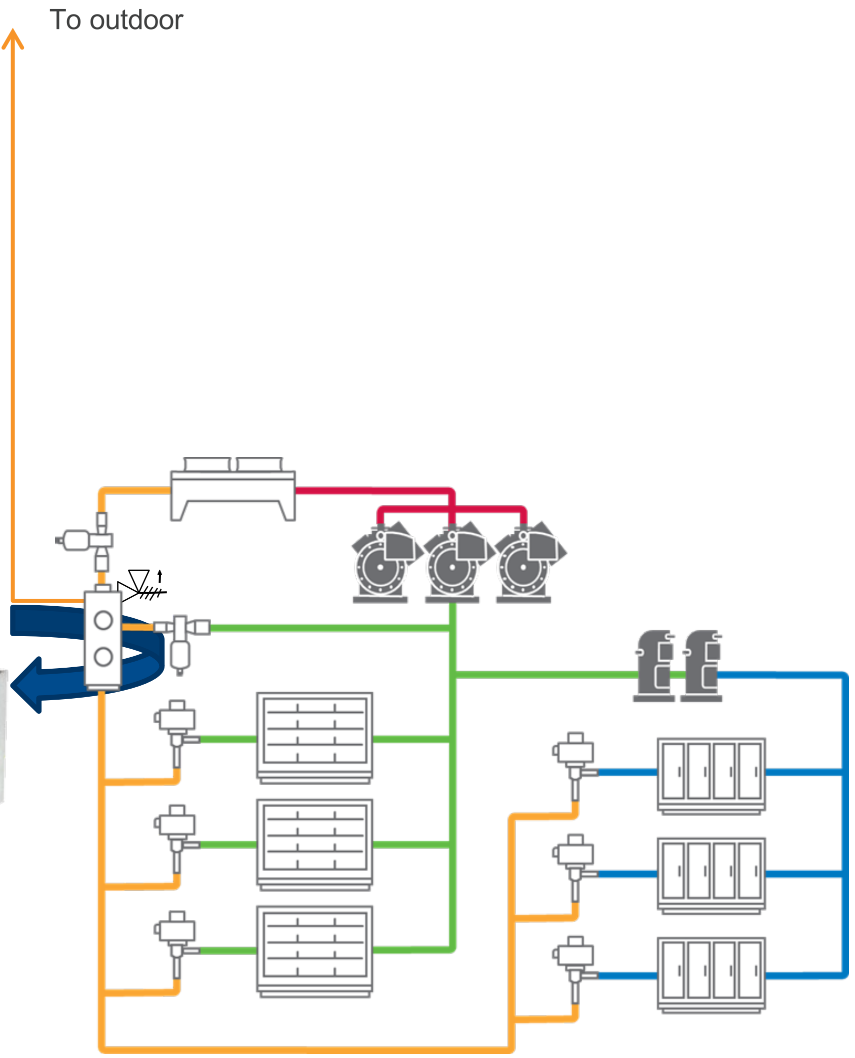
Generator/UPS source



Separate Pressure Relief System

- Auxiliary Condensing unit start on Power Failure which is Powered by Generator
- Recirculates liquid from Receiver / Flash Tank to keep the saturation temperatures below the Pressure Relief Point





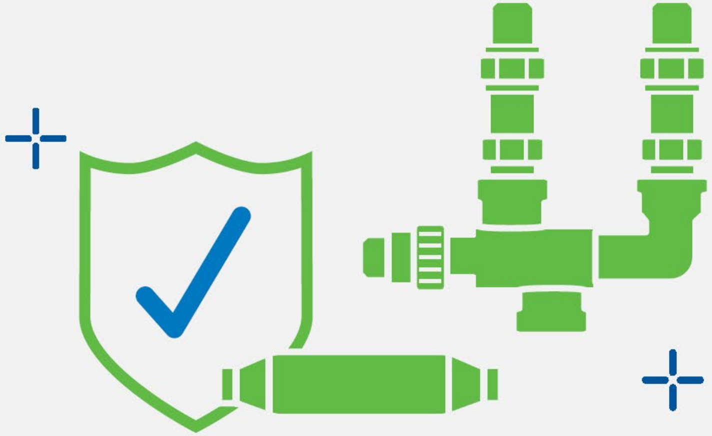
Generator Powers Auxiliary Condensing Unit

10

SYSTEM SAFETIES AND PRESSURE-RELIEF VALVES

High-pressure safety and/or mitigation strategies are primarily focused on system- and compressor-specific protection.

- Pressure-relief valves (PRVs) or check/isolation valves should be installed in various system sections to prevent a full loss of refrigerant charge.
- A system should be designed to vent its charge outdoors — per applicable codes and PRV venting standards. If the system exceeds maximum pressures, discharge lines with PRVs will safely release the charge.



Pressure Relief Valve Installation

Dual-Isolatable valve assemblies

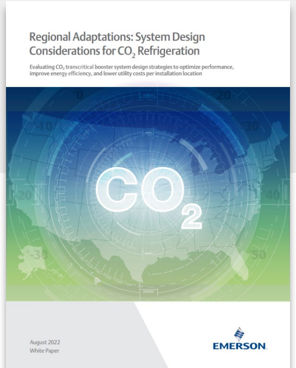
Vented outdoors

Flash Tank
650 psig (45 bar g)
2 PRVs with
3 way Valve

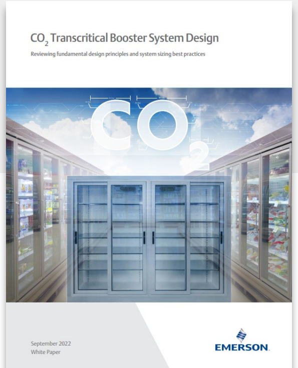
Gas Cooler
1600 psig (110 bar g)
2 PRVs with
3 way Valve

CO_2 Technical White Papers – Update and Promotional Plan

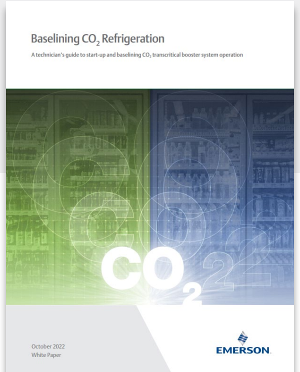
1 ***Regional Adaptations: System Design Considerations for CO_2 Refrigeration *(Not Published Yet)***



2 **CO_2 Transcritical Booster System Design**



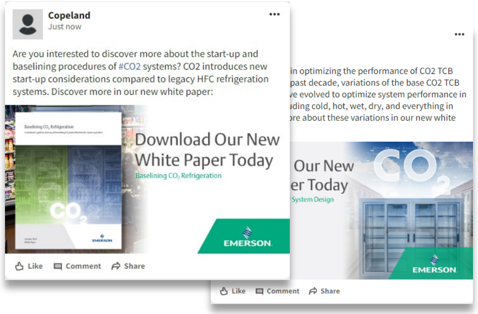
3 **Baselining CO_2 Refrigeration**



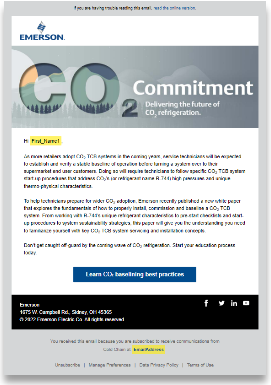
4 **Connectivity & Controls**

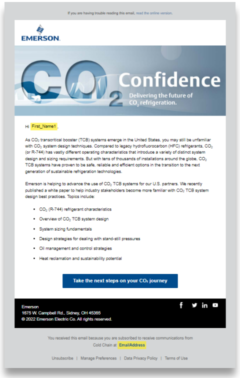
Coming soon

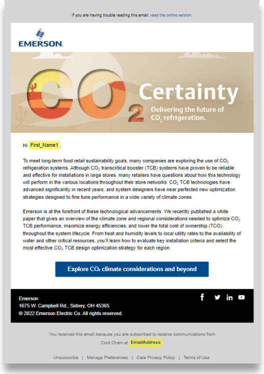
Social



Email Communications







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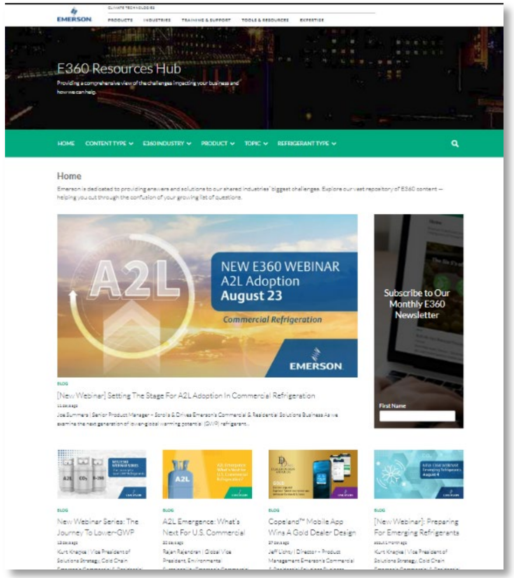
- **Product** — compressors, condensing units, facility controls and electronics, foodservice HACCP solutions, variable frequency drives
- **Topic** — connectivity, IoT and insights, contractor training, tips and safety, energy, utility and power management, innovation, maintenance and repair, refrigerant and energy regulations, sustainability
- **Refrigerant Type** — CO_2, R-290 and A2Ls



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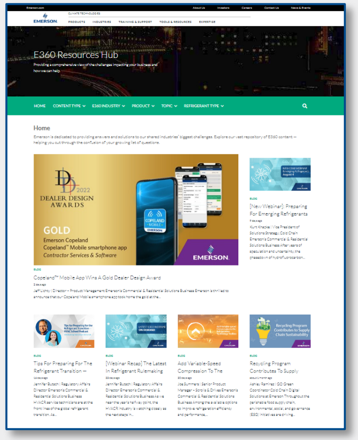

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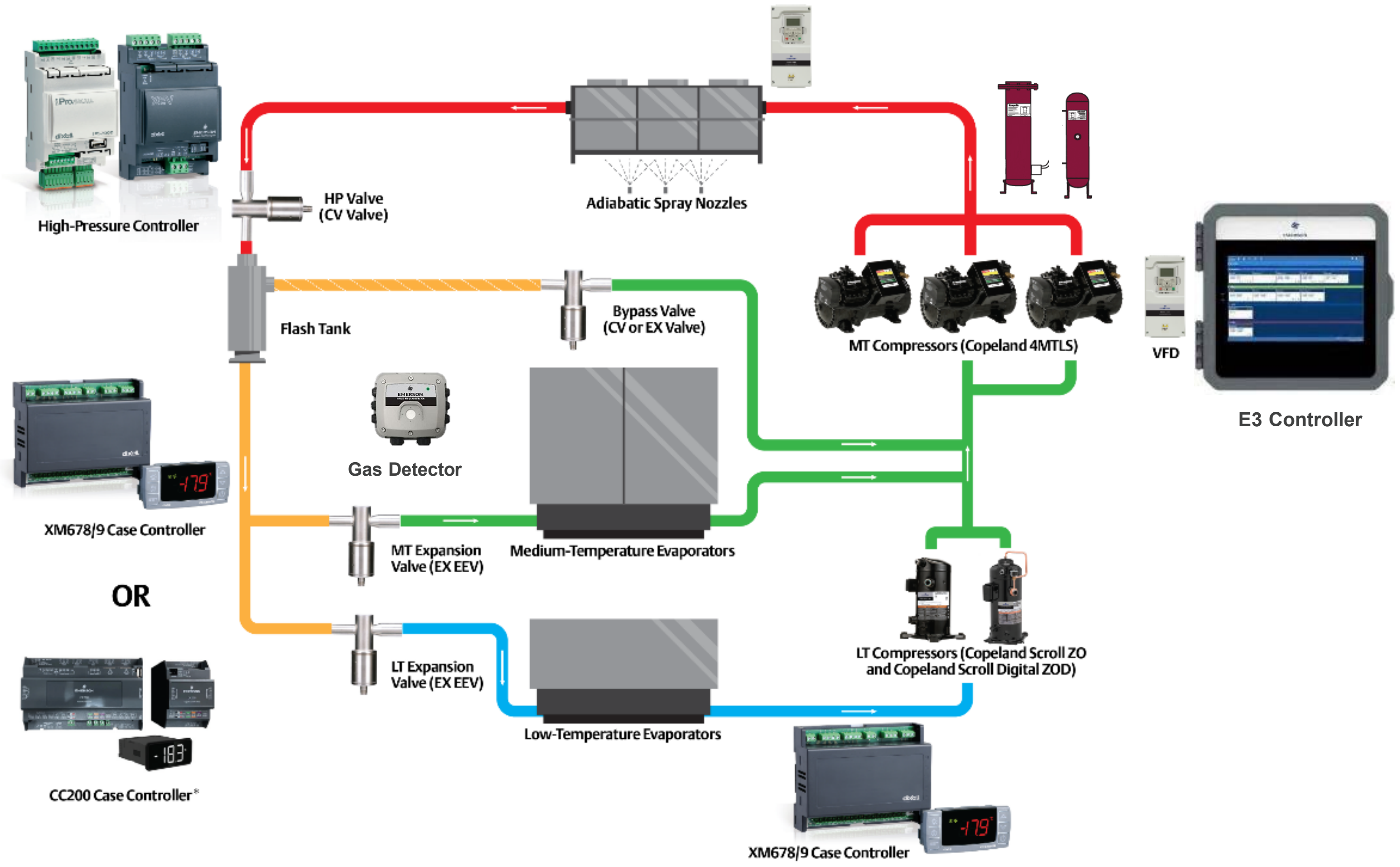
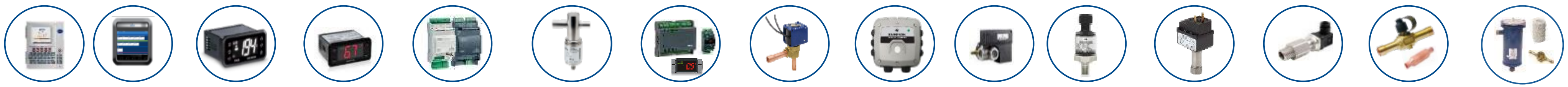
CO_2 Transcritical Booster System





E2 Rack Control | E3-CO2 Rack Control | Touch Control | Unit Control | High Pressure Controls | HPV & BPV | Case Control | PWM Valve | Leak Detection | OMC Oil Control | High Pressure Transducers | High Pressure Controls | Liquid / Oil Level Sensor | Ball / Check Valves | Filter Driers / Sight Glass

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CO2 Transcritical Booster System Operation

Presenter Name: Alain Mongrain

Emerson